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# TRUCK DESIGN OPTIMIZATION PROJECT PHASE II

PHASE I DATA EVALUATION AND ANALYSIS REPORT

> WYLE LABORATORIES SCIENTIFIC SERVICES & SYSTEMS GROUP

> > Colorado Springs Division 4620 Edison Avenue Colorado Springs, Colorado 80915



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03 - Rail Vehicles & Components

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### See FRA Report No. FRA/ORD-70/34, Phase I Data Evaluation and Analysis Plan, September 1978 16. Abstract

As part of the TDOP Phase II project, the FRA directed Wyle Laboratories to evaluate and analyze the test data acquired during TDOP Phase I for use in Phase II model validation and specification of performance indices. These data were contained on 204 magnetic tapes and computer printouts.

The applicability of the Phase I test data to Phase II was evaluated from three points of view. The first was completeness of the test matrix. Most Phase I tests were conducted using the 70 ton refrigerator car on an ASF ride control truck with new wheel profiles. Although this over-emphasis on one configuration will necessitate additional testing of the Type I truck, it was possible to derive useful information from the Phase I test data. The second was measurement accuracy. The quality of measurements was acceptable except for measurements of lateral wheel force at the wheel/rail interface and in the detection of ALD targets. The third point of view was the Phase I data's adequacy to perform the Type I truck model validation and specification of performance indices. The data in the regimes of ride quality and lateral stability appear to be adequate. In the regimes of curve negotiation and trackability, the lack of adequate measurements of wheel/rail forces makes it difficult to extract meaningful information from the data.

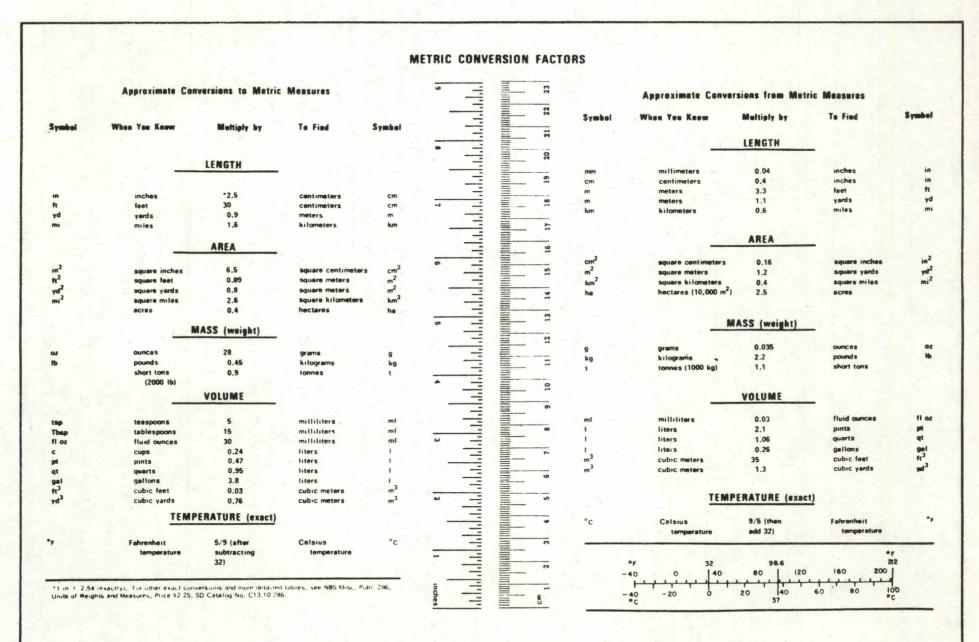
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#### EXECUTIVE SUMMARY

The purpose of this report is to evaluate and analyze the TDOP Phase I test data for its applicability to the TDOP Phase II project. Specifically, the evaluation will determine if the Phase I data can be used in Phase II model validation and performance indices specification.

The report discusses the three approaches used to determine the usefulness of the Phase I data. First, the quantity and scope of the data was evaluated. Using a data sorting routine, a series of matrices was developed. This analysis showed that the preponderance of Phase I testing was conducted on the 70-ton refrigerator car with the ASF truck and new wheels. Since the refrigerator car is not typical of most cars in service, reliance on the data may well bias the results of the Phase II analytical work.

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Secondly, the evaluation determined if the Phase I measurements accurately represent the quantity measured. For example, did the vertical accelerometers on the carbody give an accurate representation of car bounce? The conclusion was that the measurements was satisfactory except in two areas: the measurement of lateral wheel forces at the wheel/rail interface, and the detection of automatic location detector (ALD) devices. The first deficiency is of major significance. Without the data on the lateral wheel forces, the Phase I data cannot be used in validating the various curving models or in assessing the curve negotiation performance indices of the Type I truck. The lack of precise ALD target locations limits the usefulness of the Phase I data for trackability regime analysis and, to a lesser degree, ride quality analysis.

Finally, the Phase I data were evaluated for its adequacy in performing the Type I truck model validation and specification of performance indices. In other words, what data are required versus what data are available from TDOP Phase I. For the lateral stability and ride quality regimes, the data appear to be adequate; however, the lack of accurate measurements on the lateral forces at the wheel/rail interface will make it difficult to extract from the data meaningful information for the curve negotiation and trackability regimes.

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#### 1.1 BACKGROUND

As part of the Truck Design Optimization Project (TDOP) Phase II study, the Federal Railroad Administration (FRA) directed Wyle Laboratories to evaluate and analyze the test data acquired during Phase I of TDOP for use in Phase II model validation and specification of performance indices.

Prior to commencing this work, Wyle Laboratories prepared a TDOP Phase I Data Evaluation and Analysis Plan which the FRA approved (Reference 1). The plan describes what will be accomplished during the Phase I data evaluation and analysis and how the task will be implemented. It contains a description of the hardware and software to be used, the specific analytical techniques to be employed, and the selection and format of the data to be reduced. The plan also defines the expected results of this effort and the format for this report.

#### 1.2 REPORT ORGANIZATION

The remainder of the report is divided into these sections: Section 2 summarizes what data were available for this effort and the computer programs used in the study. Section 3 provides an evaluation of the data and their applicability to Phase II. Section 4 provides a sample of data usage in the form of a pilot program of data analysis for the ride quality regime. Section 5 summarizes the results of the data evaluation and analysis and provides recommendations for future testing.

#### **SECTION 2 - IMPLEMENTATION**

#### 2.1 PHASE I DATA

The TDOP Phase I data, in the form of data tapes and computer printouts of analyzed data, were provided to Wyle Laboratories by the FRA. The data were categorized by a computer-based inventory and stored in boxes. The boxes contain 204 magnetic data tapes from the five test series in Phase I, analyzed data from the car response measurements, and the track geometry. The analyzed data included the ENSCO Track Geometry Data Report, and reduced data from various test runs consisting of power spectral densities (PSDs), time histories, and statistical summaries. The complete catalog of the FRA-supplied Phase I data is contained in Appendix A.

Wyle initially explored the idea of reformatting the Phase I data to permit selection by the railroad industry of a particular phenomenon, characteristic, or parameter variation. However, a survey of the railroad industry revealed little appeal for reformatting. Furthermore, the need for a summary of TDOP Phase I data has been met by these FRA documents: the Freight Car Truck Design Optimization Phase I Executive Summary, the Test Results Reports, and the FRA Critique of the Test Results Reports (Reference 4).

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#### 2.2 DESCRIPTION OF DATA ANALYSIS SOFTWARE

#### 2.2.1 TDOP Data Sorting Routine

Because of the vast amount of Phase I data generated, a TDOP data sorting routine was developed by Wyle which provides ready access to these data. The sorting routine allows for the specification of a given set of test conditions; the routine then lists all test runs which meet that set of requirements. Details of this sorting routine and the parameters on which it sorts are contained in Appendix B. This sorting routine was used extensively in the evaluation of the Phase I data as discussed in Section 3.2.

#### 2.2.2 Post Processing Program

The analysis of the Phase I data was accomplished by utilizing the Post Processing Program developed by the Southern Pacific Transportation Company (SPTCo.). The Post Processing Program was received from the FRA on magnetic tape and converted for use on Wyle's Interdata 8/32 computer system. Documentation on the program was provided by the Post Processing Program manual (Reference 2). The effort required to convert this program to the Interdata computer system proved to be considerably more difficult than originally anticipated (see Appendix B). The program, as revised and implemented for Phase II analytical work, is described in Reference 3.

2.2.2.1 <u>Program Validation</u>. To assure the accuracy of analyzed data using the Post Processing Program, a series of steps was executed to validate the operation of the program on the Interdata computer. The first step involved running test cases on the Interdata computer and comparing results with those obtained by the SPTCo. The second step involved evaluation of the equations used in the program to determine their accuracy. The only problems experienced were in the PSD calculation. The results of the validation effort are also described in Appendix B.

2.2.2.2 <u>Enhancements</u>. The only modifications made to the Post Processing Program were those associated with the PSD package to enable it to give the correct results. These consisted of removing the mean from the signal before any PSD calculation, calculating the area under the PSD curve, removing an erroneous factor of two, and printing the gravity root-mean-square (g rms) level on the plot.

The Post Processing Program from Phase I provided plots of up to a maximum of 20 seconds in duration. A need was identified in connection with the automatic location detector (ALD) problem (see Section 3.3.2) to provide time history plots of greater than 20 seconds. This capability was implemented by writing a new program which takes the Phase I tapes and produces a reformatted tape compatible with the Wyle library of analysis routines. These routines provide the capability to produce a time history plot for one channel at a time for any duration.

#### **SECTION 3 - DATA EVALUATION**

#### **3.1 INTRODUCTION**

The data evaluation task first determined the quantity and scope of test data provided by Phase I. The trucks, the carbody types, and track conditions were identified. Secondly, the evaluation determined which measurements taken during Phase I provided useful and accurate representations of the quantity measured. For example, did the vertical accelerometers on the carbody give an accurate representation of car bounce; did the pins on which the strain gages were mounted in the adapter give an accurate representation of the lateral load at the wheel/rail interface, etc.? If the measured data did give a valid quantification of the desired parameter, they are considered acceptable for the model validation and specification of performance indices.

Thirdly, the completeness of the Phase I measurements in providing the required data was evaluated. It is not the purpose of this evaluation to judge if the data will perform the actual model validation or specification of performance indices. This determination will be made part of the analytical and engineering task areas.

The original plan for this report called for an appendix which would catalog all reduced data. However, at the completion of this task, the volume of the reduced data would have resulted in an appendix of several thousand pages. No useful purpose would have been served by publishing a report of this size. However, header sheets describing the test conditions for each run which was reduced are contained in Appendix C. All the data has been cataloged and stored at Wyle in a manner which permits ready access.

#### 3.2 DATA SORTING ANALYSIS

The data sorting routine was used to assess the number of test runs made during the Type I truck testing conducted during the TDOP Phase I test. The parameters used during this sort sequence were car type, truck type, percent load, wheel profile, and track type. The first sort is shown in Table 3-1 and shows the number of runs by car, truck and wheel type. Note that a test run in this discussion includes a number of different speeds and thus may encompass several entries in the data sorting catalog.

Table 3-1 shows that the preponderance of test runs was made with a refrigerator car on ASF 70-ton ride control trucks with new wheels. This emphasis made the data more difficult to use because the refrigerator car is not considered a typical freight car; its uneven weight distribution and very high empty weight tends to bias the data and give misleading answers. The empty weight of the 70-ton capacity refrigerator car is 89,600 pounds compared with 61,200 pounds for the empty 70-ton box car. This is approximately a 46% greater empty weight. The A-end of the empty refrigerator car weighs 49,300 pounds compared to 40,200 pounds on the B-end. This is approximately a 10% difference in the weights of the two ends. Because of these two factors, most of the analysis described in this report was accomplished using test data for the other carbody types shown.

By car type:	Refrigerator Car	234	(86%)
	70-ton Box	9	(3%)
	100-ton Box	12	(4.5%)
	89-ft. Flat	10	(3.5%)
	100-ton Hopper	8	(3%)
	Total Test Runs:	273	
By truck type:	ASF 70-ton Ride Control	225	(82%)
	ASF 100-ton Ride Control	6	(2%)
	Barber 70-ton	18	(7%)
	Barber 100-ton	14	(5%)
	ASF 70-ton Low Level	_10	(4%)
	Total Test Runs:	273	
By wheel type:	1/20 (new)	195	(72%)
	1/40 (new)	11	(4%)
Sector Const	Cylindrical	34	(12%)
	Half Worn	5	(2%)
	Worn	28	(10%)
	Total Test Runs:	273	

#### Table 3-1. Number of Test Runs by Body, Truck and Wheel Type

While the test data acquired on the test runs using the refrigerator car are considered valid, with the exceptions noted for the other data (see Section 3.3), there is a concern that using the test data from the refrigerator car may tend to skew the analytical results. As previously noted, the car's uneven weight distribution and the high empty weight can give analytical results which will not be typical for other freight cars. For this reason, Wyle decided not to include the refrigerator car in this analysis of the Phase I data. However, the dataacquired from these tests are of good quality and can be used in the analytical and engineering effort should it be required.

A more detailed breakdown by track type was conducted as shown in Table 3-2 for the refrigerator car and Table 3-3 for the other four carbody types. Again, this shows the heavy emphasis on the refrigerator car tests. On the other carbody types, only one test run was conducted for each track type. This makes any assessment of repeatability difficult.

The test data sorting information is summarized in Tables 3-4 through 3-8 which show a matrix of test combinations with a dot noting those which were tested during Phase I. Each table refers to one kind of track condition and shows the tests run by the SPTCo according to carbody type, loading condition, truck type, and wheel type and condition.

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#### Table 3-2. Number of Runs of 70-ton Mechanical Refrigerator Car

Table 3-2. Number of Runs of 70-ton Mechanical Refrigerator Car	CURVED	v / ħ	Hi-Speed Curred	Med. 50, 50, 70	Shimmer Jointed
ASF 70-ton Refrigerator Car Trucks		r			. ,
Empty, 1/20 (new) wheels	10	29	28	23	
Empty, 1/40 (new) wheels	1	1	1	2	1
Empty, cylindrical	3	4	3	2.,	3
Empty, half worn	, ,	1	2	2	
Empty, worn	1	4	4	4	
Half Full, 1/20 (new) wheels	1	2	2	.2	
Fully loaded, 1/20 (new) wheels	11	16	16	14	
Fully loaded, 1/40 (new) wheels	1	1	<u>``1</u>	1	1
Fully loaded, cylindrical	• 3	3	3	3	3
· · · · · · · ·	1	4	4	4	
Barber 70-ton trucks		_ <b></b> ,	,		,
Empty, 1/20 (new) wheels	1	1	1 :	+1	/
Fully loaded, 1/20 (new) wheels	1	1	1	2)	

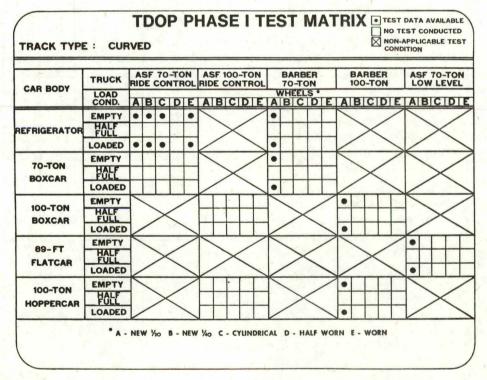
Mechanical Refrigerator)	CURVEN CURVEN	Hi-Speed	30 Speed	Med	Shi, Deed Jointed
100-ton box, Barber trucks, new wheels, empty	1	1	1	1	1
100-ton box, Barber trucks, new wheels, loaded	1	1	1	1	111
100-ton box, Barber trucks, cylindrical, empty			1	1	1
00-ton box, Barber trucks, cylindrical, loaded	And the second	1.1	at a second	1	1
70-ton box, Barber trucks, new wheels, empty	1	1	1	1	
70-ton box, Barber trucks, new wheels, loaded	2	1	1	1	
89' flat, ASF trucks, new wheels, empty	1	1	1	1	lest o
89' flat, ASF trucks, new wheels, loaded	1	1	1	1	1997
89' flat, ASF trucks, worn wheels, empty		1	1		
100-ton hopper, Barber truck, new wheels, empty	1			1.1	1999
100-ton hopper, Barber truck, new wheels, loaded	1				· · · ·
100-ton hopper, ASF truck, new wheels, empty		1	1	1	

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Table 3-4. Curved Track Test Matrix



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Table 3-5. High Speed Jointed Track Test Matrix

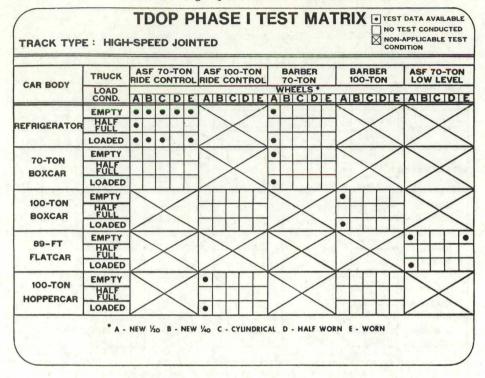
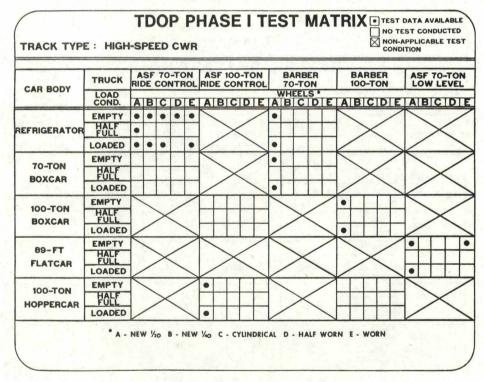


Table 3-6. High Speed CWR Track Test Matrix



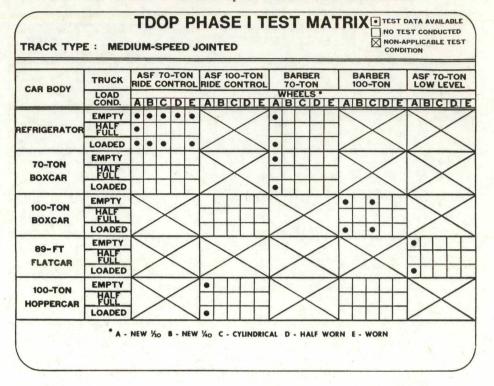
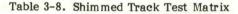


Table 3-7. Medium-Speed Jointed Track Test Matrix



CAR BODY	TRUCK			-TON		ONTROL		RBER	BAR 100		ASF 70-	
	LOAD COND.	AB	C	DE	AB	CIDIE	WHE	ELS ·	ABI		ABC	DI
REFRIGERATOR	EMPTY HALF FULL	•	•			/			$\square$	/	$\searrow$	/
	LOADED	•				/				1		/
70-TON	EMPTY					/			K	/	K	/
BOXCAR	HALE					<			1 >	$\langle \rangle$	$>$	<
	LOADED					1				1		/
100-TON BOXCAR	EMPTY HALF FULL LOADED		$\succ$	$\langle$				$\overline{\langle}$				$\langle$
89-FT Flatcar	EMPTY HALF FULL LOADED		>	$\langle$		$\langle$		$\langle$		$\langle$		
100-TON HOPPERCAR	EMPTY HALF FULL LOADED		>	$\langle$				$\overline{\langle}$				<

#### 3.2.1 Test Matrix Omissions

While it is not claimed that the test matrix need to be completely filled in for the purposes of Phase II analysis, the following omissions are considered most significant:

- a. No curving tests were run on 100-ton box cars and hopper cars with the ASF ride control truck. Since there are significant differences between the ASF and Barber trucks related to warp stiffness, a curving test should have been run with both trucks.
- b. No curving tests were run with worn wheels on any car, except the refrigerator car. Wheel wear has some effect on curving performance.
- c. The curving test runs and the conditions omitted have no significance because of the improper measurement of lateral wheel loads.
- d. No high-speed CWR tests were run with the 100-ton box car on an ASF truck, or the 100-ton hopper car with the Barber truck.
- e. No tangent track tests were run with worn wheels except for the refrigerator car, and the empty 89-foot flat car. Thus, data on lateral stability appear to be inadequate.
  - There were no medium-speed jointed rail test runs on a 100-ton box car on an ASF truck, or the 100-ton hopper car with the Barber truck. Since this type of track exercises the friction snubber, this omission makes it difficult to compare the two types of snubbing systems.
  - Shimmed track tests with other than cylindrical wheels were run only with the refrigerator car. This abbreviated test does not reflect the variety of devices present in the suspension system. An evaluation of the shimmed track tests thus requires more detailed scrutiny.

#### 3.2.2 Ride Quality Data

In terms of ride quality, the only deficiency of the Phase I data is the lack of correlation between measured track geometry and response data as discussed in Section 3.3. This should not significantly hinder the ride quality analysis as shown in the pilot program. When test data become available during the Phase II testing of Type I trucks, it can be used to further validate the results from the Phase I data.

#### 3.2.3 Impact on TDOP Phase II

The Phase I data omissions discussed in Sections 3.2.1 and 3.2.2 will necessitate additional testing during Phase II of the Type I truck. The extent of this testing will be directly related to the amount of data required by the model validation and engineering task requirements. After each of these tasks has been reviewed, a preliminary matrix of tests for the Type I truck will be prepared. These matrices will be reviewed and consolidated by the testing group and an integrated test plan developed to perform the desired tests.

#### 3.3 MEASUREMENT EVALUATION

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#### 3.3.1 Lateral Wheel Load Measurements

In Phase I, lateral wheel loads due to creep and flange forces were improperly measured. During Phase I, lateral forces between the side frame pedestals and the roller bearing adapters were measured by strain gages on pins that were located on both sides of the roller bearing adapters.

As generally known, lateral forces applied to truck components are of two types:

The first consists of external and inertial forces, such as those applied by angled couplers, centrifugal forces during curve negotiation at other than balance speed, and forces due to periodic car body accelerations having lateral components. These lateral forces are eventually reacted between the wheel and rail, and the load path passes through the bearing adapter and side frame, which justifies the method of measurement used in Phase I.

The second comprises creep and flange forces which are partly reacted between wheels of the same wheelset, and partly between wheelsets through the track structure; only the lateral components of the latter can be measured by the adapter pins on which strain gages were mounted. However, during curving with flange contact, a large part of the lateral load on the outer leading wheel is due to the creep forces on the forward wheelset; the load path is confined to the wheels and axle, thus it bypasses the adapter which transmits only the lateral creep forces from the rear wheelset (if it is not in flange contact). Therefore, lateral wheel loads measured by this method during curving at equilibrium speed are bound to be low, and the contribution of dynamic loads as coupler forces cannot be separated from those of the creep forces.

The lack of lateral wheel/rail force measurements particularly affects the curving data where the most important parameter is the lateral force at the wheel/rail interface, since this directly relates to the amount of rail and wheel wear which occurs during curving. Thus, in Phase II these missing curving tests may have to be run (hopper car with ASF truck, trucks with worn wheels), and some Phase I tests repeated to provide an adequate matrix of data to characterize the Type I truck in curve negotiation. The data available from Phase I are not sufficient for validation of any meaningful curve negotiation model since the primary quantity to be derived from the model would be the lateral force. (Also, the data do not provide sufficient information to quantify any performance indices relative to curving. However, some preliminary work can be done in the area of truck motions related to degree of curvature and superelevation.

The measurement of the lateral wheel/rail force is also of importance in lateral stability (hunting), since again it relates directly to wear. The importance of lateral force/vertical force (L/V) ratio is related primarily to dynamic regimes involving contact between the throat or flange of the wheel and the rail, either when the lateral force is high (such as occurs during hunting), or when the vertical force is low (which occurs during harmonic roll). Both situations produce a high L/V ratio and thus pose the risk of derailment.

There is no question that a better method of measuring lateral wheel loads would have been preferable than that used in Phase I. However, in the case of hunting, some very useful information may be extracted from the Phase I data by combining the vertical forces measured by the strain gages mounted in the bearing adapters with the known wheel contours, the inertial properties of the wheels and side frames, and the vertical accelerations of the pedestals to calculate the lateral loads on the wheels with a level of accuracy acceptable for engineering purposes.

Also, much model validation may be done from the Phase I data in relating critical speed to the model parameters. As several test configurations are being instrumented for curving tests, it is planned that hunting tests will be run at the same time as the curving tests with the same test configurations. Thus, some additional lateral force data will also be provided for the lateral stability regime.

#### 3.3.2 Track Geometry Correlation

A problem area discussed in the TDOP Phase I Data Evaluation and Analysis Plan (Reference 1) is the difficulty of correlating response measurements with the track geometry location. The automatic location detector (ALD) used by the SPTCo. during Phase I picked up numerous extraneous signals which made determining the exact location of the test car difficult. The technique used during Phase I for determining the exact milepost location of the test car, so that the car response data may be correlated to the track geometry measurements, was to place metal targets at known locations along the track. A detector on the train sensed the targets as the train passed over them. In theory, this method would then identify the exact location of the train; the milepost location between targets could then be obtained by integration of train speed. In practice, however, this technique did not work because the target detector also picked up extraneous signals in addition to detecting the targets.

This problem is illustrated in Figure 3-1 from test run 030201TWA001 which shows the ALD channel (solid line) versus milepost. The milepost location was obtained by integrating train speed from the known starting mile-A positive voltage signal indicates the ALD post. located a target. The dashed lines in Figure 3-1 were overlayed on the plot of ALD at those known locations at which the metal targets were placed. If the dashed lines (target location) were close to agreement with the ALD detection signal (solid lines), then it would be a relatively simple matter to put some small adjustments into the speed integration to get the dashed and solid lines to match exactly. However, the discrepancy between the two signals is so great that it is not possible to determine what corrections should be made to line them up.

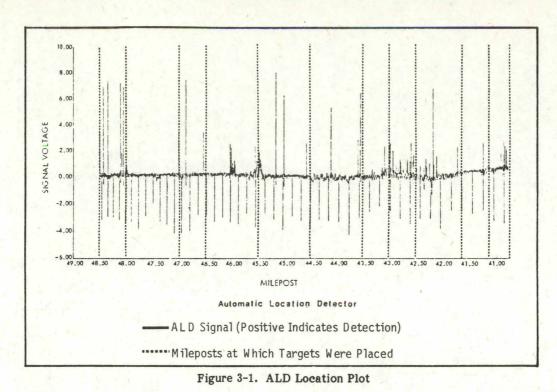
The problem of knowing the exact track input which corresponds to a given response is particularly critical in time-domain analysis. In this type of analysis, the model must be given exactly the same input as the test car if the response data are to be compared.

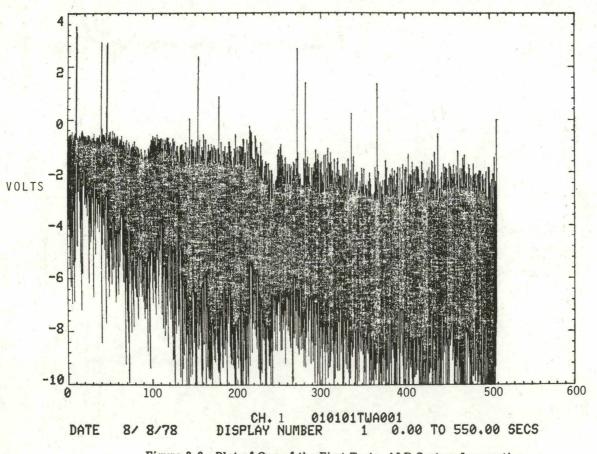
During the Phase I Data Evaluation and Analysis, the ALD signals from several runs spanning the duration of Phase I testing were plotted. Figure 3-2 is a plot of one of the first tests; the ALD system, was not operating satisfactorily at that time. After the first few test runs, the ALD signal was improved. The remaining plots show a great degree of similarity. Figures 3-3 through 3-5 all show almost exactly the same pattern. This probably indicates that some fixed object (such as a switch or crossing) causes the ALD to register and the problem that now remains is to sort these occurrences out from the actual ALD target detections. One approach to correlating the signals could be to try to relate each ALD signal with a known object and then to determine the actual ALD signals. If the ALD signal can be made to line up with the car response data, then it will be possible to use Phase I data in conjunction with the timedomain models.

At present, no additional effort is planned in attempting to correlate the track geometry and milepost location because it is not critical to perform an analysis of the data. If in the future, the track geometry/response data correlation is required, additional effort may be expended on the task.

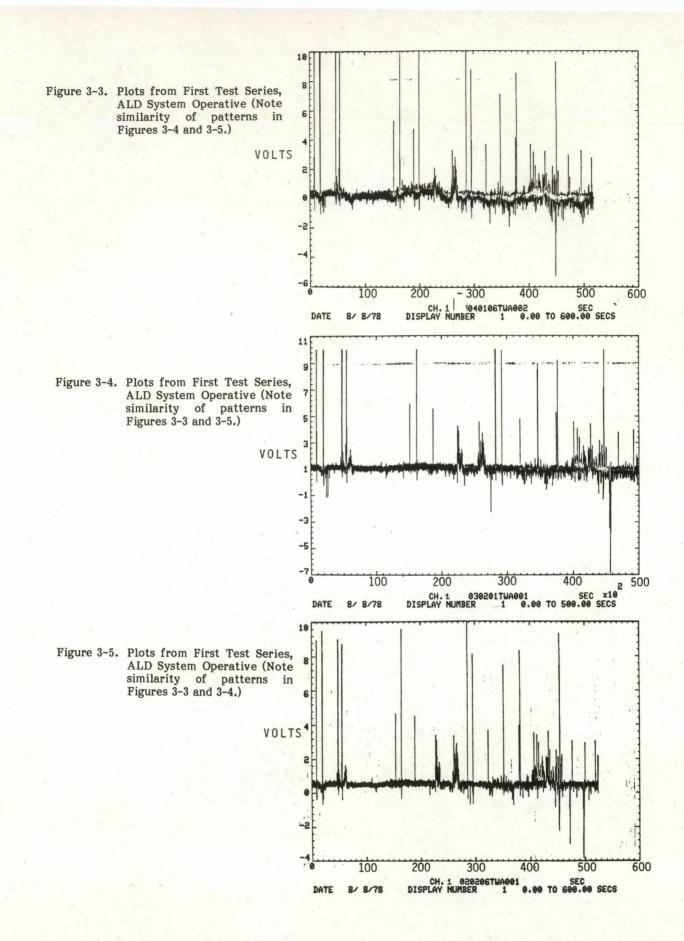
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The problem associated with the ALD was caused by the detector sensing any metal object including the desired target. This problem will be corrected during Phase II by using an alternate technique. The two techniques currently under evaluation consist of either a tuned coil or magnet buried in the ballast and an appropriate detection circuit attached to the instrumentation car. This approach should eliminate the problem of spurious signals.









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#### 3.3.3 Quality of Measurements

The measured data from Phase I proved to be of acceptable quality in the evaluation of performance parameters, with the exception of the inability of the ALD measurement to correlate to track location, the improper technique used to measure lateral wheel loads, and the lack of friction snubber force measurements.

3.3.3.1 <u>Measurements Made Incorrectly.</u> During TDOP Phase I, a strain gage mounted on a pin was used in the adapter to measure lateral force. The data acquired were accurate measurements of the lateral force at the adapter but cannot be correlated to lateral forces at the wheel/rail interface as previously discussed in Section 3.3.1. Thus, the data cannot be used in calculating L/V values. Several alternate techniques for measuring wheel lateral force are being investigated during Phase II and the most promising approach will be adopted.

The signal conditioning during the Phase I testing used a calibration technique which introduces a small error in the data depending upon the length of the cable from the signal conditioner to the transducer and the bridge resistance of the transducer. Ectron signal conditioning was used which has the excitation voltage sensing at the signal conditioner. Thus, the voltage drop it senses includes not only the transducer bridge, but also the line drop in the cable. For a 300-foot cable, this results in a two to five percent error in the calibration voltage, depending upon the transducer type. This amount of error does not significantly affect the data; however, it does introduce a slight bias on the low side to all data acquired during Phase I. Without knowing the length of cable used for each transducer on each run, it is not possible to correct for it and thus the bias is left in the data. During TDOP Phase II, the voltage sensing is being moved to the junction box on the test car. This decreases the maximum cable length to about 30 feet and the resultant error will be of a lesser order of magnitude.

3.3.3.2 <u>Measurements Not Made</u>. The lack of friction snubber force measurements was identified early in Phase I and plans were developed independently by Wyle Laboratories to design a device which will measure these forces. However, development of the device was not completed until the end of Phase I, and no over-the-road data were ever acquired. Hence, a test program using the Friction Snubber Force Measurement System (FSFMS) was conducted in Phase II to obtain the desired characterization of friction snubber forces (reference 5).

#### 3.4 APPLICABILITY OF DATA TO PHASE II

#### 3.4.1 Analysis

The Phase I data were evaluated to determine their applicability to the validation of analytical tools. This evaluation consisted of a listing, by regime, of the planned models for the Phase II analysis work, the test data requirements for each model, and the quality of the Phase I data. This survey is contained in Tables 3-9 through 3-12 for the four performance regimes planned for Phase II. Significant data are available in the lateral stability regime for model validation as shown in Table 3-9. The primary shortcomings were the lack of wheel/rail force measurements and the lack of tests on wheels with worn profiles other than for the refrigerator car. The data required for the ride quality regime are generally complete. A few minor exceptions shown in Table 3-10 were some carbody and truck motion measurements. However, these deficiencies are not considered critical.

In the curve negotiation regime, the two most critical measurements (lateral force and angle of attack) were not measured (see Table 3-11). This makes extraction of meaningful information from the other data difficult. The data available for the trackability regime is shown in Table 3-12. There are sufficient data for the linear models; however, the nonlinear models lack adequate measurement of wheel/rail forces and of correlatable track geometry.

As previously discussed in paragraph 3.2, even when adequate data channels were acquired during Phase I, the matrix of configurations which were tested is often inadequate. Thus, additional data may be required on other carbodies and wheel profiles.

#### 3.4.2 Engineering

The Phase I data were evaluated to determine their applicability to the specification of performance indices. This evaluation is shown in Table 3-13 which lists the performance index for each of the four regimes and the necessary test data required to specify the performance index. The right-hand column in Table 3-13 defines the availability of test data from Phase I for the given performance index. The data available in the ride quality and lateral stability are sufficiently complete and adequate. Limited data are available to handle portions of the trackability regime. In the curve negotiation regime, the measurements are completely lacking.

Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Engineering Models	Kinematic frequency versus speed	Measured	Satisfactory
Simple Kinematic Models	• Car body rigid body modes	Five rigid body modes available.	Longitudinal mode not measured
Eigenvalue Analysis Models	<ul> <li>Hunting at some critical speed for various wheel profiles. (Linearized models of the car/truck combination stability will be compared to the predicted critical speeds.)</li> </ul>	Hunting tests with various profiles limited to refrigerator car.	Satisfactory
Nonlinear time-domain	• Truck kinematics vs. speed	Measured	Satisfactory
models	• Car body dynamics vs. speed	Measured	Satisfactory
10 - 72 m 1	• Time histories of the vertical and lateral forces at the wheel/rail interface.	Not Measured	N/A
	<ul> <li>Mode shapes of the car/truck during limit cycle hunting motions for primary (body hunting).</li> </ul>	Measured	Limited to rigid boo modes

## Table 3-9. Lateral Stability Validation Requirements

## Table 3-10. Ride Quality Validation Requirements

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Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Linear Frequency Domain Models	<ul> <li>Nominal truck and car body vibration response data while running over tangent track on both continuous welded and jointed rail.</li> </ul>	Measured	Satisfactory
	• Duration of recorded data should be at least 100 seconds at a given constant speed in order to obtain sufficient statistical confidence in the measured PSD and transmissabilities.	Runs at most speeds averaged 60 sec. of data	Satisfactory
	<ul> <li>Track geometry shall be correlatable over the test section to within ±100 feet. (Since the work will be in the frequency domain only, the position accuracy is not as stringent as it is for the trackability regime.)</li> </ul>	Track geometry measured.	Correlation difficul Providing estimate of accuracy also difficult, +100 ft. may not be possible
	• Required truck response measurements shall include:		A Company of the
	- Vertical and lateral accelerations at each of the four bearing adapters.	Vertical accel. each end of both axles, lateral accel. on each axle.	Satisfactory
	<ul> <li>Two vertical acceleration measurements sufficient to determine vertical and roll motion.</li> </ul>	Not measured (displacement data can be used to derive roll motion).	N/A
	- Lateral acceleration measurement	Measured	Satisfactory
	• Car body		
	- Vertical and lateral	Measured	Satisfactory
	- Center A end and B end	Measured	Satisfactory
	- Both corners A and B end top and bottom	Not Measured (measurement is required to locate center of roll)	N/A
	- Lateral and vertical at car body center	Not Measured (required for flexible car bodies).	N/A

Table 3-11. Curve	Negotiation	Validation	Requirements
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Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Simple Engineering Models	Required data should provide the set up and wheel/rail forces during curving.	Not Measured	N/A
Kinematic Models	Measurements to include:		N. S. Star
	• Truck to car body yaw	Measured	Satisfactory
	Truck tram angle	Measured	Satisfactory
	Angle of attack at each wheelset	Not Measured (critical deficiency)	N/A
	• Wheel/rail forces, particularly during flanging	Not Measured (critical deficiency	N/A
Steady-State Curving	• Wheel/rail lateral force	Not Measured	N/A
Models*	Wheel/rail lateral displacement	Not Measured	N/A
Dynamic Curving Models	Data during curve entry and exit in addition to the above measurements should include time history responses of:		
	<ul> <li>Car body dynamics in the form of accel- eration measurements sufficient to deter- mine car body roll, roll center, car body yaw, sway and pitch.</li> </ul>	Measured	Satisfactory
	• Accelerometers on the truck sufficient to determine the truck component motions for:		
	<ul> <li>Truck bolster</li> <li>Side frame</li> <li>Wheelsets</li> </ul>	Lateral only measured Measured Measured	Satisfactory

\*Data acquired during the steady-state portion of the curve. (Filtering or averaging of the data will be required to extract the steady-state forces and positions.)

Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Engineering Models	Data required should be sufficient to extract truck/car resonances and mode shapes	Data available for rigid body resonances and modes.	Satisfactory
Linear Spring Mass Models	Small vehicle responses (which minimize the nonlinear reactions) over both regular and perturbed track.	Tests run on both regular and perturbed track.	Satisfactory
	Harmonic roll critical speed on perturbed track (linear models will be used as a means of estimating the critical speed before the more costly nonlinear simulations are run).	Harmonic roll available on shimmed track	Available only in two mph increments
Nonlinear Time-Domain Models	In addition to the above data, validation of the nonlinear time-domain models will require:		
	• Wheel/rail vertical and lateral forces.	Not Measured	N/A
	• Measurements prior to and during wheel lift off.	Not Measured	N/A
	<ul> <li>Extreme center plate dynamics during</li> <li>Harmonic Roll</li> <li>Bounce</li> </ul>	Measured	Satisfactory *
	<ul> <li>Truck component relative motions during perturbed track tests. (Required for validation of the large signal responses.)</li> </ul>	Measured	Satisfactory
	• All measured car truck responses shall be correlatable with the track geometry within +6 inches.	Track geometry measured	Track geometry and response data cannot be correlated.

Table 3-12.	Trackability	Validation	Requirements
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Table 3-13. Test Data Required for Engineering Analysi	Table 3-13. Test Data	Required for	or Engineering	Analysis
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Performance Regime	Performance Index	Necessary Test Data	Availability of Test Data from Phase I
Lateral Stability	• Critical Speed	Lateral Acceleration of one or more representative points on the truck measured as a function of speed and such variables as: wheel/rail contour, rail surface conditions, car bodies (truck spacing, stiffness), and lading (empty, full,)	Lateral acceleration available on axle and car body Data are taken at constant speeds of 40, 50, 60, 70 and 79 mph. Varying speeds exist between these con stant speeds. Variables such as wheel profile, rai surface conditions, car body parameters, and lading is noted in the test header. No rail contour data are available. Tests were not run for a full matrix of variables
	Magnitude of     Lateral Acceleration	Magnitude of lateral acceleration at or near the hunting speed, for the same set of variables mentioned above.	Lateral acceleration data on axles.
Curve Negotiation	• Lateral force on leading outer wheel per 1000 pounds axle load per de- gree of curve under, at and over balance speed.	Lateral force on leading outer wheel as a function of lading, degree of curvature at, under, and above balance speed.	No measurements made of lateral force.
	• Wear Index	Angle of attack as a function of lading, and degree of curvature under, at, and above balance speed.	No measurements made of angle of attack.
Alexan	• Derailment Potential	L/V ratio as a function of speed, lading, wheel/rail contour.	No measurements made from which to calculate $\ensuremath{\mathrm{L/V}}$ .
Trackability	• Wheel Unloading Index	Simultaneous loads under the wheels as a function of track twist in degrees as a function of lading.	No measurements made of vertical load at whee Vertical loads measured at bearing adapters, but can not be correlated to track geometry.
	• Max. Roll Amplitude	Max. roll amplitude as a function of excitation (amp. and frequency) for different lading conditions.	Roll angle of car body/truck bolster and roll accelera tion of car body were measured, however, they canno be correlated to track geometry.
	Rate of Energy     Dissipation	Level of friction force, displacement (i.e., spring travel), rate of increase of friction level with spring compression, as a function of lading.	No friction snubber force measurements were made.
	• Derailment Potential	L/V ratio as a function of speed, lading, wheel/rail contour.	No measurements made from which to calculate $L/V$ .
Ride Quality	• Transmissibility	Acceleration response, referred to one or more specific locations on the car body, as a function of speed, track quality and lading within the normal operating range of speeds.	Vertical acceleration made on car body. Speed, track ability, and lading were varied, however, a complete matrix of these variables was not tested.

4.

#### **SECTION 4 - PILOT PROGRAM**

Phase I data will be used as part of the analytical and engineering tasks in model validation and quantification of performance indices. To gain familiarity and confidence in the data and to demonstrate a technique for data analysis, a small pilot program was conducted during the Phase I data evaluation and analysis. This pilot study, which was intended to show how the data are to be used, was limited to one specific performance regime, ride quality.

A number of test runs were analyzed to investigate ride quality. The pilot program used rms acceleration versus speed plots to provide a visual display of the data, and regression analysis to quantify the relative magnitude of the various parameters considered during Phase I testing. A large number of variables were tested during Phase I (e.g., loading condition, carbody, rail type, wheel profiles, and truck type). This analysis attempted to address which variables had significant impact on the ride quality level.

#### 4.1 DATA ANALYSIS METHODOLOGY

For the purposes of this analysis, the rms versus speed analysis capability from the Phase I Post Processing Program was used. The Post Processing Program was given a specific series of track sections and asked to calculate the rms acceleration value. In each case, two test zones were chosen at the speed rating indicated in Table 4-1. The purpose of dividing the speed zone in half was to give some indication of the amount of spread which could occur in the results from one track section to the next.

The test runs to be considered were selected using the TDOP data sorting routine. The intent is to analyze as wide a variation of parameters as available from the Phase I testing. After much discussion, Wyle decided to concentrate on carbodies other than the 70-ton refrigerator car because there was considerable criticism of Phase I for using this test car so extensively. This limited our analysis to a workable number of runs without severely compromising the number of variables to be considered. Later in the program, the refrigerator car was included in some of the regression runs to help separate the effects of wheel profile from those of carbody types. This had little effect on the results, however.

#### 4.2 METHODOLOGY FOR SUMMARY REGRESSIONS

Wyle used a descriptive regression to summarize the results of the investigation of the TDOP Phase I data with respect to ride quality. A descriptive regression quantifies the relative effects of a number of variables. For the purposes of this study, ride quality was quantified by rms acceleration (i.e., acceleration was taken as the ride quality performance index). This was then considered a function of a number of variables such as train speed, load conditions, carbody, etc. The slope of the acceleration with respect to the train speed was estimated for speeds in the range of 30 to 79 mph. Other influences such as car loaded, car empty, jointed rail, CWR rail, etc., were represented by dummy variables, e.g., a variable whose value is either 0 or 1 depending upon which category the measurement fell into. The average change in rms level for each category was estimated. The results of this analysis are indicated in Table 4-2.

It was discovered early in the analysis that the response data measured on the axles was different from the response data measured on the carbody. This results from the fact that the axle measurements are made on the unsprung portion of the truck while the remainder of the measurements were made on the truck component and carbody which are separated from the rail input by the truck suspension system. As the rail inputs feed directly into the axle-mounted accelerometers, it is reasonable that they would have much higher accelerations than the accelerometers mounted elsewhere. It was decided to separate the axle measurements into one regression analysis and the remainder of the measurements into their own regression analysis. An analysis of variance was run with the early regressions. This analysis showed that the cross effects between the axle and the other parameters were larger than most of the primary effects which strongly suggested that this was an appropriate division to make.

It is important to note that a descriptive regression does not attempt a curve fit of the data. Individual curves could be fitted using the least-square techniques, each curve having a separate equation. Individually fitted curves would provide a more accurate representation of the data. However, information regarding the relative size of the effects would be obscured. Since the purpose of this analysis is to determine the relative importance of the various parameters, we have chosen to describe the data with the regression, obtaining an indication of the average size of each effect. Similarly, the equations used do not force the acceleration to go through zero when the train speed is zero. The equations should be regarded as linear approximations to the "real" function in the range of the train speed variable considered (e.g., 30 to 79 mph).

The following example clarifies the use of the data in Table 4-2. Suppose it were desired to estimate the rms acceleration level in the lateral direction at the A-end roof of a fully loaded 70-ton box car traveling at a train speed of 40 mph on CWR with new wheels and a Barber truck. The total rms acceleration is calculated by adding the rms acceleration contribution of each of the variables, as shown in the following equation:

g rms acceleration = g rms/mph x speed + g rms (acceleration location) + g rms (loading) + g rms (carbody) + g rms (rail type) + g rms (acceleration direction) + g rms (wheel profile) + g rms (truck type) + constant rms acceleration = .00172 g rms/mph x 40 mph + .0609 g rms + 0.0 + .0189 g rms - .0186 g rms - .0185 g rms + 0.0 + 0.0 - .0188 g rms = .0927 g rms.

This predicted value of .0927 g rms based upon the regression analysis may be compared to measured test values taken on the above configuration of .0728 g rms and .0702 g rms. The error here is typical, 68% of the data may be expected to have an error within  $\pm$  .0329 g rms.

However, the importance of the analysis is not so much a quantitative prediction of the g rms levels, but a qualitative prediction of how the variables affect the measured g rms level. While it is an accepted practice in the railroad industry to report ride quality as an rms level, this is not necessarily appropriate for all modes of deterioration. Rms is an <u>average</u> level. It may be that certain types of lading can accept an rms level of around 1 g rms, but suffer damage if some peak accelerations is exceeded (e.g., 10 g peak).

Table 4-1. Track Sections

	Mile P	Post Numbers
Speed-mph	Jointed	CWR
30	48.5 - 48.25	42.5 - 42.75
30	48.25 - 48.0	42.75 - 43.0
40	47.75 - 47.5	43.25 - 43.625
40	47.5 - 47.25	43.625 - 44.0
50	46.75 - 46.38	44.5 - 45.0
60	46.38 - 46.0	45.0 - 45.5
60	45.0 - 44.75	46.94 - 47.37
70	43.75 - 48.38	42.5 - 43.38
70	43.38 - 43.0	43.38 - 44.25
79	41.6 - 41.15	45.5 - 46.44
79	41.15 - 40.7	46.44 - 47.37

Note: Samples per zone: 3000 to 9300

## Table 4-2. Summary Regressions

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	Measurement Not On Axle	Measurement On Axle
Slope of Speed	.00172 + .00008 g rms/mph	.00376 + .00025 g rms/mph
Accelerometer Location	the part of the state of the st	
Axle	N/A	Nominal
Truck Side Frame	.0884 <u>+</u> .0062 g rms	N/A
Roof of Car	.0609 ± .0064 g rms	N/A
Car Center	.0029 _+ .0061 *g rms	N/A
Center Sill	Nominal	N/A
		Street and a second
Empty as Opposed to Loaded:	.0287 <u>+</u> .0085 g rms	No Data
Car Body		
70-ton box	.0189 <u>+</u> .0045 g rms	.0242 <u>+</u> .0107 g rms
89-ft flat	0004 <u>+</u> .0079 *g rms	No Data
100-ton Hopper	0115 <u>+</u> .0105 *g rms	No Data
100-ton box	Nominal	Nominal
CWR as Opposed to Jointed:	0186 + .0041 g rms	0496 <u>+</u> .0107 g rms
Lateral as Opp. to Vertical Accel:	0185 <u>+</u> .0062 g rms	1561 <u>+</u> .0107 g rms
Worn as Opp. to New Whels:	.0131 <u>+</u> .0105 *g rms	No Data
ASF as Opp. to Barber Truck:	0013 <u>+</u> .0105 *g rm.	No Data
Constant:	-0.188 <u>+</u> .0066 g rms	.1109 <u>+</u> .0164 g rms
R <sup>2</sup> **	72.6%	82.8%
Std. Error	.0329 g rms	.0545 g rms
Number of Samples	338	104

\*Cannot be distinguished from nominal at 5% significance level.

\*\*Ratio of explained variance to total variance.

To demonstrate what information may be extracted from Table 4-2, consider the following: as the speed goes up, so does the g rms acceleration level; the empty car has a rougher ride than the loaded; the 100-ton hopper gives the best ride; wheel profile and truck type have too small an effect to be distinguished from zero, based upon these data.

#### 4.3 PRIMARY INFLUENCES ON RIDE QUALITY

The primary influence on ride quality, as measured by rms acceleration readings from the TDOP Phase I data, was train speed. As the train speed increased, the g rms level increased. Another major difference in the mea-surements was the significantly higher g rms levels measured on the truck axle, as opposed to measurements made elsewhere on the truck and carbody. Train speed was expected to play a major role in determining ride quality. As the train moves faster, there is more kinetic energy available to excite the car. Thus, one expects the accelerometer readings to increase roughly as the square of the train speed. Similarly, the distinction between measurements on the car and measurements on the truck were expected because the truck is designed to cushion the car from the rail. The unsprung mass at the wheelsets should respond more violently than the much heavier carbody. The difference in level between axle and car measurements merely indicates that the truck is operating as expected.

1.

The effect of train speed on ride quality is clearly visible throughout the data. This is shown in Figures 4-1 through 4-4 where rms acceleration is plotted against train speed. An rms value is plotted for the first and second half of each speed zone. The true data are represented by the symbols. The lines connecting the symbols are for visual clarity only, and are not intended to represent any information at other speeds. Figures 4-1 and 4-2 show vertical and lateral acceleration as a function of train speed for travel over jointed rail and Figures 4-3 and 4-4 are for travel over continuous welded rail (CWR). The expected trend may be seen in each of the figures where the rms level tends to increase with speed. In particular the effect becomes more pronounced as the measurement is taken at locations closer to the rail. However, it tends to be obscured by a resonance phenomenon (e.g., buildup due to rocking at 50 mph in the 100-ton box car). This caused problems in estimating a squared relationship in the summary regression. A least square curve fit of the data in Figure 4-1 will tend to bow down because of the resonance. Without the resonance points in the data, the curve would tend to bow up, which is the desired effect. Thus, it was decided to use a straight line approximation instead of a least squared curve fit.

The cushioning effect of the truck is also illustrated in Figures 4-1 to 4-4. In Figure 4-1, the highest rms accelerations are shown to occur on the axle, with lower levels occurring on the side frame, and the lowest levels on the carbody itself. This is also indicated in the summary regression from Table 4-2. As mentioned earlier, data measured on the axle were separated from data measured elsewhere to obtain a more accurate representation. The marked differences between coefficients in these regressions indicate the size of distinction in the data, e.g., a slope of .00376 g rms/mph for data measured on the axle compared with .00172 g rms/mph for the rest of the data. The ratio between vertical and lateral accelerations on the axle is quite different from that on the carbody. The lateral acceleration is a smaller proportion of the vertical acceleration on the axle than on the carbody. The distinction between CWR and jointed rail is larger numerically but is roughly the same proportion in the axle data as in the other data, and the distinction in carbodies is even smaller in the axle data than in the other data. Finally, the accelerometer on the side frame of the truck reads higher (.0884 g rms as shown in Table 4-2) than any of the other locations considered in the carbody regression. This suggests the extent to which the truck succeeds in cushioning the car.

Considering the car alone, the major influences on rms ride quality seem to be speed, and the distinction between empty and loaded cars. The level of the measured rms is dependent on the location of the transducer. The highest rms levels were measured on the axle with significantly lower levels being measured elsewhere on the truck and the carbody.

Empty and loaded rms acceleration plots in Figures 4-5 and 4-6 for both jointed and CWR show the empty car to have consistently higher levels. There seems to be little difference between the average level on jointed versus CWR. Accelerations on the roof averaged 0.0609 g rms higher than accelerations at the A-end center sill or at the center of the carbody on the floor of the car. No significant distinction was found between the center sill and the center of the car indicating that the carbody was fairly rigid, e.g., flexible modes of the car do not play a major role in these data. On the average, empty cars rode rougher than fully loaded cars by 0.0287 g rms. This was expected when the mass of the system decreases (i.e., the car is empty) the acceleration must increase if the force causing the motion does not decrease in proportion. Another interpretation is that the friction snubbers are sized for fully loaded cars; hence, they over-damp the empty cars.

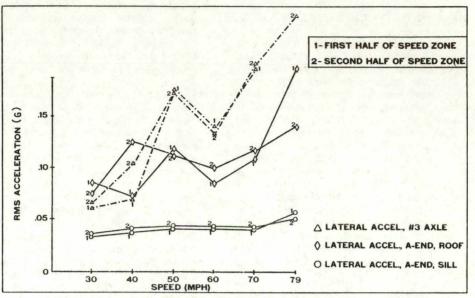
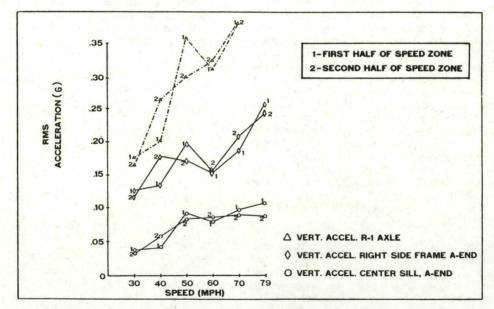
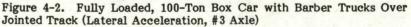
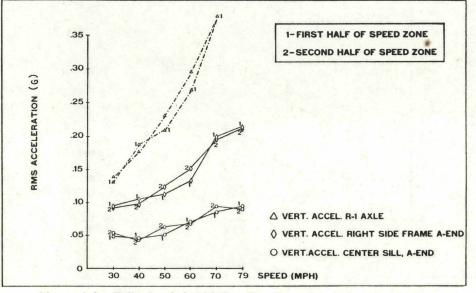


Figure 4-1. Fully Loaded, 100-Ton Box Car with Barber Trucks Over Jointed Tracks (Vertical Acceleration, R-1 Axle)







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Figure 4-3. Fully Loaded, 100-Ton Box Car with Barber Trucks Over CWR Track (Vertical Acceleration, R-1 Axle)

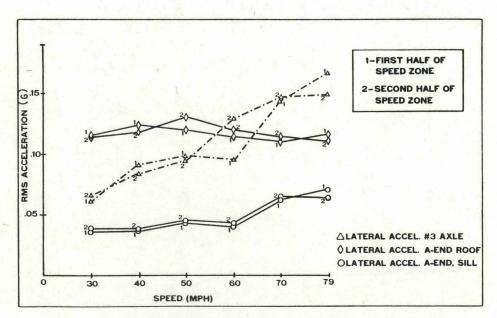
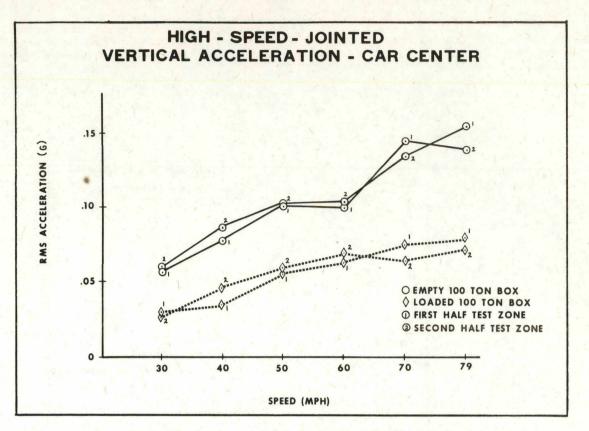


Figure 4-4. Fully Loaded, 100-Ton Box Car with Barber Trucks Over CWR Track (Lateral Acceleration, #3 Axle)



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Figure 4-5. Empty vs Loaded rms Acceleration Plots (Jointed Track)

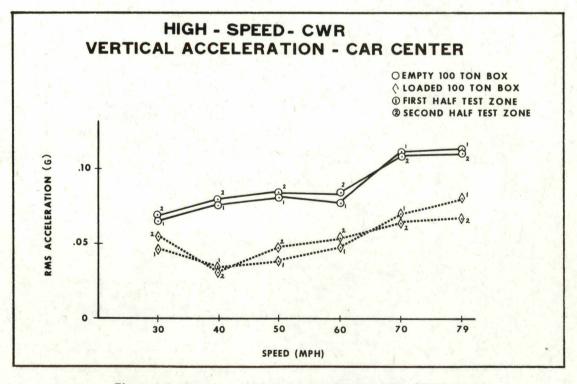


Figure 4-6. Empty vs Loaded rms Acceleration Plots (CWR Track)

#### 4.4 SECONDARY INFLUENCES ON RIDE QUALITY

Most of the distinctions (carbody type, wheel profile, rail type) investigated had only a secondary influence on the ride quality. These are shown in Figures 4-7 through 4-9 which compare plots of rms acceleration for new vs. worn wheels, Barber vs. ASF truck, and 100-ton vs. 70ton box cars. In each of these cases, the difference in rms acceleration is less than in previous plots. In particular, the type of carbody, the type of rail, and the accelerometer orientation all exhibited about .019 g rms effects. Regardless of the truck manufacturer, new wheels did not exhibit any influence that could be distinguished from zero at 5% confidence level.

Four carbodies were investigated: the 100-ton box car, the 70-ton box car, the 89-ft. flat car and the 100-ton hopper car. Only the 70-ton box car was significantly different from the other cars (averaging .0189 g rms more than the others). Interpreting the results for the 89-ft. flat car is compounded by the lack of data taken on trucks similar to the ones used in the other tests.

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Only the ASF low-level truck was run under the 89-ft. flat car, and this truck was not run under any other carbody. Data for the 89-ft. flat car in a loaded condition have not been considered to date (where flexible behavior might be expected). The similarity of the results for the different carbodies tends to suggest the cars were behaving rigidly.

The rail type (i.e., CWR or jointed rail) showed the expected effect: the CWR averaged .0186 g rms less than the jointed rail. This supports the hypothesis that the joints are one of the causes of the excitation.

Similarly, laterally oriented accelerometers averaged .0185 g rms less than vertically oriented accelerometers. This suggests that most of the motion excited from the rail is vertical (at least in the ride quality regime).

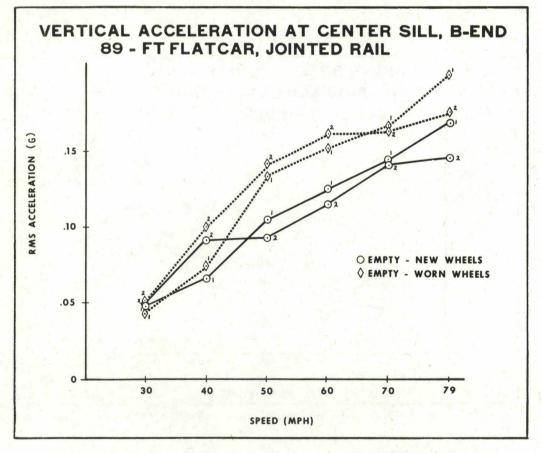


Figure 4-7. Comparative Plots of rms Acceleration (New vs Worn Wheels)

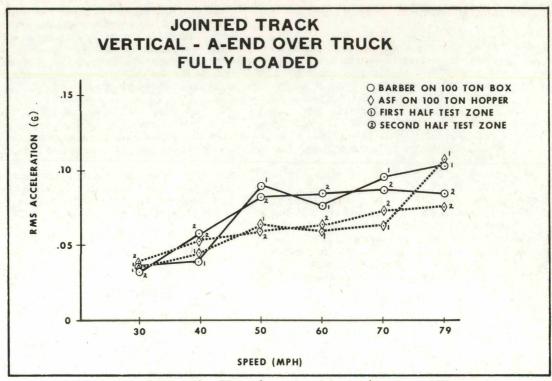


Figure 4-8. Comparative Plots of rms Acceleration (Barber vs ASF Truck)

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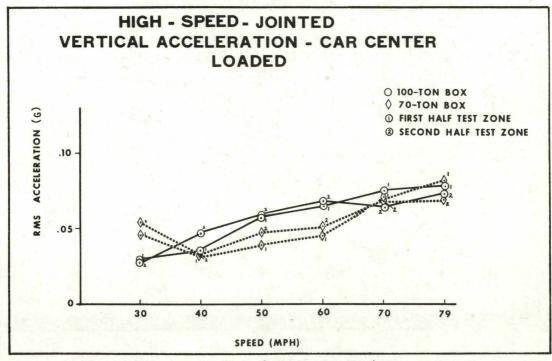


Figure 4-9. Comparative Plots of rms Acceleration (100 ton vs 70 ton)

#### 4.5 RELATION TO ECONOMICS

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Based upon the size of the measured effects (level of g rms), it seems doubtful that the levels and distinctions being reported here are large enough to have a major effect on lading damage (i.e., differences in the .02 g rms range probably are too small to play a major role in lading damage). The one exception is speed. From this analysis, it appears that operational considerations (train handling, humping, etc.) and sensitivity of the lading probably play a larger role in lading damage that the items discussed in Table 4-2. This is not to say that resonant phenomenon like harmonic roll or instabilities like hunting are not important in determining lading damage; however, for this pilot program, the concentration was on ride quality, rather than on these other performance regimes.

Another interpretation of these results is that the performance index selected (acceleration) does not measure the source of the problem. For example, the peak acceleration levels might be quite different from the rms. To assess this, the preceding analysis was rerun with peak acceleration and average absolute amplitude as performance indices rather than rms. Naturally, this changed the numbers obtained. However, the ratio between the numbers did not change significantly. Essentially, rms accleration was as good a prediction of the size of differences between carbodies (for example) as peak acceleration. One exception was observed during the course of this analysis, but it was traced back to an accelerometer with insufficient sensitivity.

#### **SECTION 5 - CONCLUSIONS**

After acquiring the Phase I data, necessary changes were made to the SPTCo software (the Post Processing Program) to make it operational on the Interdata computer. The task of modifying the Post Processing Program to run on the Interdata computer proved to be quite difficult. The sorting routine that Wyle developed made it possible to obtain an accurate idea of what was available in terms of test configurations from the Phase I data. While there were many gaps in the available data as noted in this report, it was still possible to derive useful information from the data. This was demonstrated by the pilot program for the ride quality regime.

#### 5.1 APPLICABILITY OF DATA TO PHASE II

The applicability of the Phase I test data to Phase II was evaluated from three points of view. The first was completeness of the test matrix. To determine this Wyle developed the TDOP data sorting routine; the results of this routine are shown in the series of matrix tables in Section 3.2. This analysis showed that the preponderance of the testing was conducted on the 70-ton refrigerator car with the ASF truck and new wheels. Wyle believes that the refrigerator car was not typical of most cars in service and that using these data from these measurements in any extensive manner might tend to bias the results of the analytical work. Thus, these data were not used in the pilot program and may be used only sparingly in the analytical and engineering tasks. Furthermore, some significant configuration combinations are missing from the test matrix.

The second manner in which the Phase I data were evaluated was from the point of measurement accuracy; how well did a given combination or set of channels reproduce the desired measurement parameter? In all areas except two, the quality of measurements was acceptable. These two unacceptable areas were in measurement of lateral wheel force at the wheel/rail interface and in the detection of ALD targets. In particular, the lack of lateral forces at the wheel/rail interface is of critical importance to TDOP Phase II. Without it, there is little that may be done in validating curving models or assessing the curve negotiation performance indices on the Type I truck. Also, these missing data will have a secondary influence on the analysis of lateral stability because the time-domain models cannot be validated. The lack of ALD target detection (not being able to correlate ALD targets with response) limits the usefulness of the data for analysis of the trackability regime. The lack of ALD correlation hampers the ride quality evaluation to a lesser degree.

The third point of view was in the Phase I data's adequacy to perform the Type I truck model validation and specification of performance indices. In paragraph 3.4, the required data vs. available data from Phase I is shown. The data in the regimes of ride quality and lateral stability appear to be adequate for the Phase II effort. In the regimes of curve negotiation and trackability, the lack of adequate measurements of wheel/rail forces makes it more difficult to extract meaningful information from the data.

In summary, the one critical flaw with the Phase I data was the lack of lateral force measurements at the wheel/rail interface. This deficiency will require correction via additional testing of the Type I truck during TDOP Phase II.

#### 5.2 USAGE OF DATA DURING PHASE II

This evaluation and analysis study was conducted to determine the applicability of data acquired during Phase I to the analytical and engineering effort being conducted on TDOP Phase II. The results of this applicability were discussed in detail in Section 3. The usage of data will be addressed in the analytical and engineering task efforts.

The data analysis routines in the Post Processing Program were reviewed and corrected so that correctly analyzed data will be obtained from the data analysis.

#### 5.3 RECOMMENDATIONS FOR FUTURE TESTING

The critical lack in Phase I data of lateral and vertical force measurements at the wheel/rail interface must be corrected in TDOP Phase II. The primary goal of testing on the Type I truck during Phase II will be to measure these forces. The first step will be to conduct an extensive study of available techniques for measuring these forces and to prepare recommendations for a technique to be used during Phase II. The second step will be to develop the transducers necessary to provide the required measurements. To measure the angle of attack of the wheels relative to the rail, we plan to use displacement transducers which will measure the relative position of the wheel and rail. To provide a positive correlation between track geometry and carbody response data, an ALD system will be developed which will explicitly determine the test car location relative to the track. The ALD systems will consist of a buried magnet or tuned coil and a detector system on the instrumentation car which will sense the field of the buried target as the car passes over it. By placing the ALD system prior to starting Phase II testing and by using the same ALD system on all testing (track geometry, friction snubber, Type I truck, and Type II truck), it will be possible to correlate all measured data taken during Phase II of TDOP.

To complete the test matrices, test carbodies will be tested, using new and worn wheel profiles. The primary concentration of Phase II testing on Type I trucks will be in the curve negotiation performance regime. However, tests will be conducted as well for the trackability and lateral stability performance regimes. Data for the ride quality may then be extracted from the other three regimes. These tests should be run on vehicle configurations already prepared and instrumented for the curve negotiation tests and should not require any additional preparation time. The purposes for running the additional trackability and lateral stability tests are:

- a. To complete information not previously obtained during Phase I (i.e., 100-ton hopper car on the ASF truck and a hopper car with worn wheels).
- b. To provide some degree of continuity between Phase I and Phase II data (by repeating one or two Phase I runs, a comparison may then be made between data from the two programs).
- c. To provide a final validation check of models (i.e., models validated using the Phase I data may be used to predict Phase II test results).
- d. To provide test data over yard track to assess the ability of the truck to traverse severe changes in track configuration.

#### REFERENCES

- 1. FRA Report No. FRA/ORD-78/34, "TDOP Phase I Data Evaluation and Analysis Plan," September 1978.
- FRA Report No. FRA/ORD-78/12.XIII, Volume 12, "TDOP Post Processing Program Manual," February 1978.
- 3. Wyle document TDOP TR-OX, "User Operator's Manual for Post Processing Program Operation on Interdata 8/32," May 1978.
- 4. FRA Report No. FRA/ORD-78/12, "Freight Car Truck Design Optimization, Volume I, Executive Summary; Volume VI, Critique of Phase I - Test Series Results Reports; Volumes VII through X, Results Report for Test Series 1 through 5," February 1978.
- 5. FRA Report No. FRA/ORD-79/24, "Friction Snubber Force Measurement System Field Test Report," August 1979.

#### APPENDIX A

#### INVENTORY OF PHASE I DATA

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DESCRIPTION LOCATION	DESCRIPTION LOCATION	.DESCRIPTION LOCATION
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LOCATION	80X 21	80X 22	R0X 22	R0X 22		BOX 23	BOX 23	ROX 24											BOX 24				
. 0ESCR1P110N	MAM RUISUN (EASTBOUNN) Scheivylle Br (EAStanunn) Utput Papp July 75 Tack nata Scheivyle 70 Decements	MULES LINE (R.N., WEARDIND) NILES LINE (EASTROJUN 80X 22 75 TRACK DATA Duitput Papep July 75 TRACK DATA Emb suislun(MESTBOLUND) MMM XISLUF(EASTROLUND)	NILEA LINE(MESTBOUND) Output Paper May 75 track nata Eastanumn Niles Line	W.B.WTLFS LINE Utput paper track geometry measurfments fra 4113014	MAR ATTEND SHELLVILLE RANCH (R'R HESTBOUND) Lohbad Shemard thark(r.R. Westround) Niles Line (Eastround)	DUTTUIT PAPER LOAD TWA 40 MPH ZONE DW DTSK	DITPUT PAFFD MESTBOUND MATHITME. Suitsum-Bahai High Spfr Dittput Paper Load Twa 50 mph 20NE	HAX 24 SFDIES 1-1-X MUTPHIT PAPED 010104TWA002 010104TWA001	010104T#4003 010104T84001 010104T84001 010104T84001	010104T5R001 010104T5R001 010104T5W001	0101101555001 0101105755001 0101105755600 01011057556004 01011047556004	0101n;TEW01 0101n;TEW01 0101n;TEW01 0101n;TEW01	0101ATEM001 0101ATEM001 0101ATEM001 0101ATEM001 0101ATEM001	0101ATE4001 0101ATE4001 0101AATE4001 0101AATE4001	010104TW4001 010104TW4001 010105TW4001 010105TW4001	010144744003 01014604601 0101461601 01014564601		01010100000000000000000000000000000000	00% 25 NUTFHT PAPER 1-1-%/1-2-%	01170117 56755 010261754001 010261754001 0102607544001	010204TE4001 010204TE4001 010204TE4001 010204ATE4001	010245754001 010245754001 010245754001	
	c	E C	· ·	•		c	6 6	EC															
LOCATION	R0X 17	Bnx 17 Bnx 17	BOX 17	90% 17 0	ROX 17	B0X 17 0	ROX 17 0	E C	80% 17	ADX 17	RIX 18 80% 18	B0% 18	BIX 18	RDX 18	ROX 18	BOX 16	19 FNSTO TRACK DATA Interdated Exceptions, curvature Adx 19 And Ded Intar Reduction Report Test orjective Test or-97,2	BOX 20	ROX 21	R0X 21	12 XU8	RNX 21	

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A-4

LOCATION	1E X06	35 Xur		80X 33
DESCRIPTION	CCIFCCC LCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC	SERIFS 3 STAC PAC CURVES 099557194 099557194 099557194 099557194 099557194 099557194 09955719400 099557194001 0402017414002 04020147144002 04020147144002 04020147144002 04020147144002 04020147144002 0402147144002	0.0101011174 0.001011154 0.000071154/H 0.0000710500 0.001010500 0.001010500 0.001010500 0.0000105144 0.0000105144 0.000007144 0.000007144 0.00007101144 0.00007101144 0.00007101154/H0002 0.001017154/H0002 0.001017154/H0002 0.001017154/H0002 0.001017154/H0002	90X 33 0117417 PAPER SERIES 0003101764 0=1/2/3/4=X 0401017744 040101744 040101744 040101744 040101744 040101744 040101744 040101744 040207744001 040207744001 040207744001 040207744001
LOCATION	- 90X -	ROX 29		rurvrs
. DESCRIPTION	0203047EM 0203047EM 020357TEM/H 020357TEM/H 020357TEM/H 020357TFM/H 020357TFM/H 020357TFM/H 020357T5M 02037T5M 02037T5M 02037T5M 02037T5M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M 02037T7M			0177017 PAPER SCL FIRVE 030201794/R 030201794/R 030201794/H 0302017744/H 0302017744/H 0302017744 030201743 030201744 030201743 030201744 0302000000000000000000000000000000000
LOCATION		80 X 26	ROX 27	
. DFSCRIPTION	102011400 102011400 102011400 102011400 102011400 102011400 10201400 10000000000	DUTFUT PAFFS DUTFUT PAFFS 010247144001 010247144001 01024744001 01024744001 01024744001 01014744 010147400 01024744 01024744 010247440 010247440 010247400 010247400 0102474000 010147600 01000 000000	010107526600 0101076600 010101606 010101606 958763 4-3-4 958763 4-3-4 00101778/4 020101778/4 020101778/4 02010179/4 020101794 020101794 020101794 020204794 020204794 020204794 020204794 020204794	00000000000000000000000000000000000000

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	DESCRIPTION	LOCATION	DESCRIPTION	LOCATION	DESCRIPTION	LÖCATION
		********				
	040201 TEM/H					
	040202TEM/H	· ·	0405n7CNE 0404n3CNE		0501n4T8M001	
	0402nTTEM/H		040502CND		0501n3T8M001 0501n1T8M001	
	040201TEM/H 040201TWA002		040403CNO 040502TSM/R		0502naTSH001	-
	0402n2TWA001		040403T3M/R		050205TSM001 050203TNA001	
	0402n3TWA001 0402n4TWA001	2	040406TEM/H 040501TEM/H		050203TWA001	i.
	040301 TWA001		040501CNR	•	050205THA001 0502n1M0D	e e
	0403n9TWA001 0403n3TWA001		040404CNR 040404CNR		050203M0D	
	0403ngTWA001		USUDALCOR	1	050201TSM 050205M0D001	
	04020118M/R 04020218M/R		OUTPUT PAPER	80X 35	05020STEA	· · · · ·
	040203TSM/R		040301TWA 040205TWA		050204TEA 050204TEA	· ·
	04020ATSM/R 040301TSM/R	,	040302TWA		050203TFA001	
	0403npTSM/R	1	040203TWA		050205TWA001 050305TEA001	
	0404nj TWADO1	1	040101TWA		0503njTEA001	
	0404n7TWA001 0404n3TWA001		010201TEM 040304TWA		050302TEA001	
	DADANATHA -		040301 TWATO1		0503037EA001 - 0503037EA001	r
	040405TWA ~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~		040302TWA001 040303TWA001		050305TEA001	
	0404nøCNE	,	0403001WA001		050303CND001 050303CNE001	i
	040405CNE	-	050101M0D		0503n2CN0001	
	0405n1CNE		0501n2MDD 0501n1TE4002	•	050302CNE001	. 2
	040404CNE	· · · · · · · · · · · · · · · · · · ·	0501natEA001		BOX 38 SERTES 5-3-X DUTPUT PAPER	
	0404nd <b>13</b> M/R 0404n <b>s1</b> 5M/R		050103TEA001 050101TWA002		DUTPHT PAPER	90x 38
	0404neTSM/R		050103TWA001	·	0503n17MA001 0503nn7WA001	· •
	0404nj15M/R 0404n719M/R	1	0501naTWAD01	,	050305TWA001	•
J.	0404nTTSM/R	· ·	0501ngM0D001 0501n1TWA002	, ,	050302TWA001 050301TEA001	ſ
	0404naTEM/H 0404nstem/H		050102TEA001		- 050302TEA001	
	040406TEM/H		050101TEA002 040101TEM002		/ 0503naTEA001	
	040401 TEM/H001		050103CME001		0503n5TEA001 0503n1TSM	
	040402TEM/H001		0501ntCN0001 0501ntCN0001		050302TSM	
٠	0404n1CN0	<i>r</i>	050101CN0001	· 	0503n#78M 0503n#7WA001	· ·
	9494n7CNN 9494n7CND		OUTPUT PAPER		0503naTFA001	~
	0494naCN0 -	• • • •	050101TSM003	BOX 36	0503n3M0D001 0503n1 T8M	
	1404n5CN0 7404n6CH0		05010100000	80X 36	0503n2 TSM	
	DADANICNE		OUTPUT PAPER	80X 36	- 0503nt TSM	1
	04040PCNE	*	050101M0D002 /		0503ng 18M 0503ng 18M	5 %
	040403CNE /	4.	050104M0D001 050103M0D001	BOX 36 BOX 36	050302CNE001	× ,
SERI	ES 2 BOOK & TIME DOMAIN GRAD		OUTPUT PAPER		0903n7CN0001 0503n3CNE001	,
	NEW MHEELS REDUCED SPRING WITH N=5 SPRINGS	T M G	050103M0D001	80X 36 ~ 80X 36	0503n3CN0001	
	,		514CMF001	BOX 36	050301CNE001 050301CN0001	
SEN1	E8 5 BOOK 1 70-TON CAR HIGH JDINTED RAIL 05010XTWA 050	-SPEED BOX 34	514CN0001	BOX 36	05030 M00001	· , · · ·
			OUTPUT PAPER	80X 36	050303 THA 050302 THA	
SERI	ES 5 ANNK 2 70-TON CAR HIGH Welden Rail Osoioxtea Osoi		SERIFS 5 TO TOM CURV	/FS	050301 TWA	
			OUTPUT PAPER	80x 36	050403 TWA 050400 TWA	,
SERI	ES 5 BOOK 3 70-TON CAR MODIA TRACK 05010XMOD 05020XMOR	TEDBOX_34	010101CN0 		050965 THA	•
			050202CND		050301 TWA001	· · · · · · · · · · · · · · · · · · ·
SERI	ES 5 BOOK 4 70-TON CAR MEDIL		0502n3T84001	• •	050303TWA001 050302TWA001	
	JOINTED RAIL 05010XTAM 050	JEUATSM	0502ngTSM001 0502n5TSM001	· 4	0503n1TEA001	~ •
OUTP	UT PAPER 0403nstem/H	; BOX 34	0502n1CNE	, ·	050304CNE001 050304CNE002	
	0403n4TEM/H 0401niTFM/H	1	0502n1CNR 0502n1CND	·, ·	050303CN0001	
	0403n#TEM/H	· · · ·	05020ACNE		0503n1CN0002 0503n1CNE001	
	040102TEM/H 040301TEM/H	4	050205CNR 050208CN0 / ×			· · · · · · · · · · · · · · · · · · ·
	D403n2TEM/H	× ,	050201M0D		REPORT NO. FRA-OR+D 75-818 16 Coptes	`BOX 39
	0403n#TEM/H 0403n#TEM/H		050203M0D 050203TEA	· .	DUTPUT PAPED SERIES 5-3/4-X	80X 39
	040201TEM/H	-	050203TNA		050301784	
	040202TEM/H		05020TEA		05030278M 05030378M	,
	040203TEM/H 04020aTEM/H		050204TEA 050205TEA		0503naTSM /	
,	010201TSR		05020740001		050305T8M 050305M0D001	×
	0405#2TSR 0405n1TSR		0502n4M0D001 0502n5M0D001	*	0503naM00001	
	040501T8M			,	SPRING COMPARISON EMPTY 050301CN0001 RDX TSM	
	010101T8R 010101T8M	,	OUTPUT PAPER 0502nttea	80X 37	050303TWA001	·
	0401njTWA		050201TEA		050302MDD	
			0502n1 TVA - 0501 n1 TWA		0503n3M0D	а. А.
	0405n1CN0 0404n6CN0	-	- 0501011NA 050101TWA		050301MDD	
	040502TEM/H	,	050104TMA		050302MDD 05030178M	
,	040463TEH/H 040167TFM/H		05020XTWA 050200THA	· _	, 8503ng18M	
	0401ALTEM/H		050205TAA	•	. 050305T8M 050602TWA001	
			· · ·	, '	· · · · · ·	*
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A-6

DESCRIPTION	LOCATION		OCATION
0506n2TEA001		EMPTY TEST 030202	
050401 TEA001 050401 TWA001 050405CN0001		SERIES 3 BANK 5 50-FT 70-TAN BOX CAR LOADEN TEST 030301	80X 50
050402TWA 050402TWA		SERIES 3 BOOK 6 50-FT 70-TON BOX CAR	80X 50
050405TWA 050301TWA 050302TWA		EMPTY TEST 030302 BERIES 3 BOOK7 100-TON COVERED HOPPER	80X 50
050305TWA		LAADEN TEST 030401	
RDX 40 OUTPUT PAPER SERIES 5-4-X Dutput Paper 0504nstsm001	BOX 40	SERIES 3 BANK & 100-TON COVERED HOPPER Empty test 030402	BOX 50
050403TSM001 050404TSM001		SERIES 3 BOOK 9 89-FT FLAT CAR LOADED TEST 030502	BOX 51
0504naM0D001 0504naM0D001 0504n1M0D001		SERIES 3 BOOK 10 89-FT FLAT CAR EMPTY TEST 030501	BOX 51
050402TWA001 050402TEA001		SERIES 2 BOOK 1 RMS.HISTOGRAMS AND	80X 52
0504n5CN0001 0504n5CNE001 0504n5CNR001		PSD'S SERIES2 BOOK 2 TIME DOMAIN GRAPHS	BOX 52
050405TEA001		AVERAGE WORN WHEELS (FMPTY)	
050401TEA001 050402TEA001 050405TWA001		SERIES 4 ROOK 5 CURVED RAII FMPTY TESTS 040X0YCNE,040X0YCNO	BOX 52
050407TFA001 050401THA001		SERIES 4 BOOK 4 CURVED RAIL LOADED TESTA:040X0YCNR,040X0YCNE	80X 52
050405TEA001 050403TEA001 050403TEA001		040X0YCN0	BOX 52
0504naCNR001 0504n3CNR001		SERIES 4 RAAK 3 MEDIUM SPEEN JOINTEN Rail tests:040x0ytsm.040x0ytsr	00A 35
050405CHR001 050403CHR001 050403CHE001		SERIES 2 BOOK 3 TIME DOMATH GRADHS WORN WHEELS D=5 SPRINGS (LOADED)	BOX 53
050403M0D001 050405TWA001 050400CN0001		SERJES 2 BOOK 4 TIME DOMAIN GRAPHS MORN WHEELS D=3 + D=7 SPRINGS (LOADED)	BOX 53
050404MD001 050404MD001 050405M0D001 050405M00001		SERIES 2 BOOK 5 TIME DOMATH GRAPHS WORN WHEELS D=5 SPHINGS (FMPTY)	BOX 53
050407TWA001 050405TWA001 050405T8M		SERIES 2 BOOK 6 TIME DOMAIN GRAPHS WORN WHEELS D=3 + D=7 SPRINGS	POX 53
0504na15M 0504na15M 0504n515M		(EMPTY) Series 2 Rook 7 Time Domain Graphs New Wheels,D=5 Springs,2/3 Snubbing	80X 53
050402TSM 050401TSM 050403TWA001		(LOADED AND EMPTY) SERIES 3 BOOK 11 89-FT FLAT CAR EMPTY	BOX 53
050403TEA001 050403CN0001 050404CN0001		WORN WHEELS TEST 030503 Series 4 Book 1 High Speed Jointed	BOX 53
050405CN0001 050403CNE001		RAIL TEST 040X0Y TWA	
050404CNE001 050405CNE001		SERIES & BAAK 2. HIGH SPEED WELDED Rail Testi040x0ytem, n40x0ytem	
SERIES 1 ORTGINALS	BOX 41	SERIES 2 GRAPHS BOOK B	BOX 54
SERIES 2 PINTS TIME DOMAIN 8.5X11	BOX 42	SERIES 5 GRAPHS BOOK 1=4	BOX 54
SERIES 3 PINTS, REEFER, BOX SCL Series 3 Pints Flat Scl	BOX 43 BOX 44	SERIES 5 ROOKS 70-TON CAR CURVED TRACK Over-FG, Speed 05010xCN0	BOX 55
SERJES 4 PI NTS	80x 45	0502nxCN0	
SERIES 3 PINTS REEFER, BOX SCL		BOOK & TO-TON CAR CURVED TRACK EQUILIBRIUM SPFFD 05010xCNE	
SERIES 5 PINTS 8.5 X 11	80X 46	OSOSCATCHE	
SERTES 2 OPTGINALS	BOX 47	BOOK 7 70-TON CAR CURVED TRACK RESONANT SPEED	
SERIES 5 TIME PLOTS, ORIGINALS SERIES 1 BOOK 1 LOADED REFERS	BOX 48	05020VCNR BOOK A 100-TON CAR HIGH-SPEED	
SERIES 1 MOOK 2 HALF LOADED	BOX 49	JOINTED RAIL 05030xTWA	
SERIES 3 POOK 1 MECHANICAL REFRIGER- ATOR I DADED TEST 030102 ROOK > MECHANICAL REFRIGER- ATOR I CADED TEST 030101 ROOK 3 60-FT 100-TON ROX CAR I CADED TEST 030201	BOX 49	05040XTWA BOOK 0 100-TON CAR HIGH-SPEED Welden Ratl 05030xTEA 05040XTEA	
SERJESS BOANA GO-FT 100-TUH BOX CAR	ROX 50	ROCK TO 100-TON CAR MODIFIED TRACK 050304000	

	DESCRIPTION	LOCATION
	05040×MOD	
	and the second of the second	
SERTES	S BOOK 11 100-TON CAR MEDIUM Speen Jointed Rail 05030xtsm 05040xtsm	80X 56
SERIES	S BOOK 12 100-TON CURVED TRACK NVER-FO, SPEED 05030xcN0 05040xcN0	BOX 56
SERIE	5 ROOK 15 100-TON CAR CURVED TRACK EQ. SPEED 05030XCNE 05040XCNE	80X 56
SERIES	S BOOK 14 100-TON CAR CURVED TRACK RESONANT SPEED 05040XCNR	80X 56
CLEAN	TAPES.	80X 100
	16249-77-2-40	
	16249-77-2-38	
	16249-77-2-30	
	16240-77-2-44	
	16249-77-2-42	
	16249-80-2-39	
	16249-80-3-31	
	16249-80-3-29	
	16249-80-3-49	
-	DRAWT-UP R	
TOOP	DRAWINGS	80X 57
	ROLL	
	ROLL & SCL BOX CLASS X-5-8	
	ROLL C R-70-24	
	ROLL N 8-100-33	
	ROLL F L + N HOPPER	
	ROLL F F-70-65	
	ROLL & 70 TON ASE RIDE CONTROL	
	ROLL H TO TON BARBER S-2 STABLIZED	
	POLL T 100 TON BARBER S-2 STARLIZED	
	LOW PROFILE	
	ROLL , 70 TON ASF LOW LEVEL	
	ROLL K 100 TON ASF AS RIDE CONTROL Roll I TIMKEN MOD + STD ADAPTERS	
	70 + 100 TON	
BOX 1	16249-67-3-15	80X 102
	16249-67-3-05	
	16249-69-3-38	
	16249-67-3-13	
	16249-67-3-11 16249-69-3-42	
	16249-69-3-40	
	16249-69-3-46	
	16249-69-3-44	
	16249-67-3-17	
	TAPES.	BOX 103
	16249-77-1-36	HUA 105
	16249-77-1-50	
	16249-77-1-30	
	16249-77-1-28	
	16249-77-1-32	
	16249-77-1-34	
	16249-77-1-20	
	16249-77-1-1 16249-77-1-16	
BLANK	TAPER	BOX 104
- Seite	16249-80-4-31	THE AVE
	16249-80-4-35	
	16249-77-1-14	
	16249-80-4-25 16249-80-4-27	
	16249-77-2-14	
	16249-80-4-17	
	16249-80-4-21	
	16249-80-4-47 16249-80-4-09	
BLANK		
	TAPER. 16249-80-4-37	

	16249	-77-3-18
	16249	-77-3-49
	16249	-77-4-48
		-77-4-41
		-77-3-43
		-77-4-34
		-77-4-46
		-77-3-10
		-77-3-09
LANK	TAPES	
		-69-2-46
		-69-2-44
	16249	-69-2-42
		-69-2-40
		-69-1-10
		-69-1-16
		-69-1-46
		-69-1-20
		-67-3-20
		-69-2-30
1011		/ENDJOB

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DESCRIPTION

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LOCATION

BOX 105

#### PHASE I DATA SORTING ROUTINE

Wyle Laboratories developed a data sorting routine to provide a ready access to, and analysis of, Phase I data. A subsequent upgrading consisted of bringing the sort program parameter into agreement with the information contained in the Phase I Final Report, (1) and in the magnetic data tape headers. The tape header for those runs that the data were reduced are contained in Appendix C.

The sorting routine permits the user to specify a given set of test conditions; the program then lists all test runs which meet that set of requirements. The program sorts on the nineteen parameters or test conditions listed in Table B-1. Shown below each parameter in this table are the possible variations of the parameter and the user code which is specified for a search on that parameter. Any combination of the user codes may be used when making a search. However, many combinations will produce a null set. As an example, a sort for 100-ton box car and 70-ton ASF ride control truck will produce a sort with no entries.

An example of a typical sort printout is contained in Table B-2. This was a sort for all tests which were run over high-speed, jointed track on a fully loaded 100-ton box car, with a Barber 100-ton truck equipped with new wheels. The result was one test run consisting of six entries, one each at 30, 40, 50, 60, 70 and 79 mph. The Test ID, inventory box location, and the tape number are contained in the first two lines. With this information, it is possible to determine the required tape and retreive it for data reduction. This sorting program was used extensively in the evaluation of the Phase I data.

# POST PROCESSING PROGRAM

The Post Processing Program developed by SPTCo. was used to analyze the Phase I data. It was received from the FRA on magnetic tape and converted to run on Wyle's Interdata computer. Documentation for the program was provided by the Post Processing Program manual.(2) The effort required to convert this program to the Interdata computer proved more difficult than originally anticipated. The original program was supplied in EBCDIC and required conversion to seven-bit ASCII for the Interdata computer. The assembly language subroutines were completely rewritten using the Interdata assembly language and one subroutine was modified to work with the ASCII decimal equivalents rather than the EBCDIC decimal equivalents. To get the Post Processing Program to fit on the Interdata computer, it had to be overlayed and the number and size of plots that could be requested was reduced to 10 plots with 20 lines.

<sup>(1)</sup>FRA Report No. FRA/ORD-78/12.II, "Freight Car Truck Design Optimization Volume II, Phase I Final Report," February 1978.

<sup>(2)</sup>FRA Report No. FRA/ORD-78/12.XIII, Volume 12, "TDOP Post Processing Manual," February 1978. The original manual for the program was revised (3) to include the changes that were made for operation on the Interdata computer. It contains samples of all files used in building the load module, a sample request deck, and the output from the request deck.

Some problems were also experienced in the original version of the program and have been noted. These problems were discovered during the course of reducing data and show that the program is not yet fully checked out. For example, at least one plot must be requested as there are many FORTRAN do-loops which use the total number of plots as the upper limit, and zero is not an acceptable value. Also, at least one set of equation cards must be included, even if the data are not required, because a zero equation is unacceptable to the program logic. Also, the original manual was not clear as to where blank cards must be positioned nor where they may be detected. These problems were eliminated in the revised version of the program.

SPTCo. recommended that at least 450k bytes of memory core be available for operation of the Post Processing Program. This total did not include space for such things as the operating system or any enhancements to the system, which would rule out use of the program without extensive modification on any machine with less than 512k bytes of core. Problems were also encountered by going to a non-IBM system, e.g., file structures were different, routines required rewriting, the EBCDICto-ASCII conversion was necessary, etc.

Difficulty may be encountered even when trying to install the Post Processing Program on an IBM computer. The manner in which the program handles peripheral devices, such as I/O and disk storage, would make its use on any IBM machine difficult, unless the system had a nearly identical set of peripherals to that for which the program was written. For example, the program was written to output to three separate line printers, and would require modification if a lesser number of line printers were available.

As previously discussed, a program manual is available from NTIS for running the Post Processing Program. While this manual was an invaluable guide in making the program operational on the Interdata computer, anyone attempting to use it should be aware of certain errors and unclear passages in the manual, as discussed in this report. Some of these have been noted in this report, however, not all analysis combinations were tried and some errors may still exist.

## **Program Validation**

To assure the accuracy of data analyzed by using the Post Processing Program, Wyle executed a sequence of procedures to validate the operation of the program on the Interdata computer.

<sup>(3)</sup>Wyle Document TDOP TR-0X, User Operator's Manual for Post Processing Program Operation on Interdata 8/32," May 1978.

# Table B-1. Sort Program Parameters

TAPE NO.			D6	USER CODE = 13,
	WHERE N = TAPE NO.	USER CODE = 1, N,	D7	USER CODE = 13,
			D8	USER CODE = 13,
CARBODY				
	70 TON M RFR	USER CODE = $2, 1,$	SNUBBING (OUTER)	1. 1.8
	100 TON 60' BOX	USER CODE = $2, 2,$	ASF 3091	USER CODE = 15,
	70 TON 50' BOX	USER CODE = $2, 3,$	2/3 NORMAL	USER CODE = 15,
	89' FLT CAR	USER CODE = $2, 4,$	B 432	USER CODE = 15,
	100 TON C HPR	USER CODE = 2, 6,	ASF 3221	USER CODE = 15,
	attended to a start of the		B 421	USER CODE = 15,
TRUCK TY			B 422	USER CODE = 15,
	ASF 70 TON R.C.	USER CODE = 3, 1,		
	ASF 70 TON L.L.	USER CODE - 3, 2,	SNUBBING (INNER)	
	ASF 100 TON R.C.	USER CODE = 3, 3,	B 433	USER CODE = 16,
	BARBER 70 TON	USER CODE = $3, 4,$	ASF 3222	USER CODE - 16,
	BARBER 100 TON	USER CODE = $3, 5,$	ASF 3092	USER CODE = 16,
	and shine in the offering a second	A LAN THE MAN	8442	USER CODE = 16,
TRUCK CEI		NEW WORK ALL THE	B 422	USER CODE - 16,
	45' 9''	USER CODE = $4, 1,$	B 433	USER CODE = 16,
	46' 3''	USER CODE = $4, 2,$	3091	USER CODE - 16,
	41' 3''	USER CODE = $4, 3,$		
	40' 10''	USER CODE = $4, 4,$	SNUB. AUG.	
	64' 0''	USER CODE = $4, 5,$	NO AUGMENTATION	USER CODE = 17,
		Complete Martin State	VOLUTE	USER CODE = 17,
PER CENT	LOAD		HYDRAULIC	USER CODE = 17,
	EMPTY	USER CODE = 5, 1,	TRUCK CEER AUG.	USER CODE = 17,
	HALF FULL	USER CODE = 5, 2,	and the second second second second second	
	FULLY LOADED	USER CODE - 5, 3,	C PLT. FRICTION	
			STEEL-MOLY	USER CODE = 18,
HEEL PRO	DFILE		COMP. STEEL	USER CODE = 18,
	1-20 (NEW)	USER CODE = 6, 1,	STEEL-STEEL	USER CODE = 18,
	1-40 (NEW)	USER CODE - 6, 3,		
	CYLINDRICAL	USER CODE = 6, 3,	FILE NO.	
	HALF WORN	USER CODE = 6, 4,	N	USER CODE = 19,
	WORN	USER CODE = 6, 5,	WHERE N = FILE NO.	
	UTER SPG			
NU. UF U	N N	USER CODE = 7, N,	TRACK	
	WHERE N = NO. OFSPGS.	USER CODE - 7, N,	CURVED	USER CODE = 20,
	WAEKE N - NO. UFSPOS.		SHIMMED	USER CODE = 20,
	(OUTED)		HI SPD JTD	USER CODE = 20,
SPG ITFE	(OUTER)	USED CODE - 0 1	HI SPD CWR	USER CODE = 20,
	D1	USER CODE = 9, 1, $USER$ CODE = 0, 2	MED SPD JTD	USER CODE = 20,
	D2	USER CODE = 9, 2,	and the second second second	
	D3	USER CODE = 9, 3,	SPEED	
	D4	USER CODE = 9, 4,	N	USER CODE = 21,
	D5	USER CODE = 9, 5,	WHERE N = SPEED	
	D6	USER CODE = 9, 6,	and the second	
	D7	USER CODE = 9, 7,		
	D8	USER CODE = 9, 8,	OUTER GIB CLEARANCE	
			1/4 OGC	USER CODE = $22$ ,
NO. OF I	NNER SPG	a station in the second second	5/8 OGC	USER CODE = $22$ ,
	N	USER CODE = 11, N,	a second second second second second	
	WHERE N = NO. OF SPGS.		SIDE BEARING	
			3/8 SB CLR	USER CODE = 23,
SPG TYPE	(INNER)		1/4 SB CLR	USER CODE = 23,
	D1	USER CODE = 13, 1,	5/8 SB CLR	USER CODE = 23,
	D2	USER CODE = 13, 2,	1/8 SB CLR	USER CODE = 23,
	D3	USER CODE = 13, 3,	2.5 K PRELOAD	USER CODE = 23,
	D4	USER CODE = 13, 4,	5. K PRELOAD	USER CODE = 23,
	D5	USER CODE = 13, 5,	7.5 K PRELOAD	USER CODE = 23,

Table B-2. Sort Example

TEST CONDITIONS.	CAR TYPE 1	OO TON 60' BOX US	ER CODE = 2, 2,		
TEST CONDITIONS.			ER CODE = 3, 5,		
TEST CONDITIONS.			ER CODE = 6, 1,		
TEST CONDITIONS.			ER CODE = 5, 3,		
TEST CONDITIONS,		I SPD JTD US	ER CODE = 20, 3,	NUMBER OF ENTRIES =	6
ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA003
BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 38
FILE NO. 1	FILE NO. 1	FILE NO. 1	FILE NO. 1	FILE NO. 1	FILE NO. 1
TAPE NO. 89	TAPE NO. 89	TAPE NO. 89	TAPE NO. 89	TAPE NO. 89	TAPE NO. 8
100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX
BARBER 100 TON	BARBER 100 TON	BARBER 100 TON	BARBER 100 TON	BARBER 100 TON	BARBER 100 TON
46' 3''	46' 3''	46' 3''	46' 3''	46' 3''	46' 3''
FULLY LOADED	FULLY LOADED	FULLY LOADED	FULLY LOADED	FULLY LOADED	FULLY LOADED
1-20 (NEW)	1-20 (NEW)	1-20 (NEW)	1-20 (NEW)	1-20 (NEW)	1-20 (NEW)
7-D50/7-D5I	7-D50/7-D5I	7-D50/7-D5I	7-D50/7-D51	7-D50/7-D51	7-D50/7-D51
8-B432	8-B432	8-B432	8-B432	8-B432	8-B432
8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS
NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION
STEEL-MOLY	STEEL-MOLY	STEEL-MOLY	STEEL-MOLY	STEEL-MOLY	STEEL-MOLY
HI SPD JTD	HI SPD JTD	HI SPD JTD	HI SPD JTD	HI SPD JTD	HI SPD JTD
30 MPH	40 MPH	50 MPH	60 MPH	70 MPH	79 MPH
5/8 OGC	5/8 OGC	5/8 OGC	5/8 OGC	5/8 OGC	5/8 OGC
1/4 SB CLR	1/4 SB CLR	1/4 SB CLR	1/4 SB CLR	1/4 SB CLR	1/4 SB CLR

This first step in these procedures was to run the test case described in the SPTCo. documentation for the program. The test run used was 050101TWA002 and the results are documented. The results agreed exactly with those the SPTCo obtained except the Interdata computer plotted only two time-history plots on a page versus six for the SPTCo case.

In the second step of the validation process, Wyle compared independently developed time histories to verify conversion of raw, multiplexed data (in volts) to a correct time history in engineering units. During the initial TDOP Phase II proposal effort, Wyle independently developed software for a limited analyses of the TDOP Phase I tapes from NTIS. A five-second, timehistory plot was made, using an in-house program, which is shown in Figure B-1 for each speed zone run during the test. The measurements used consisted of channels 5 and 9. These same channels and corresponding mileposts were run using the Post Processing Program and are shown in Figures B-2 to B-6. The starting time for each of the plots in Figure B-1 is noted on the corresponding plot in Figures B-2 through B-6.

The two curves agree exactly in amplitude indicating that the conversion to engineering units was made correctly. However, when comparing the response at a specific milepost between the two time histories, a difference is noted in that the two time histories show a shift in the milepost at which a particular event occurs. As shown in Table B-3, the milepost comparison grew progressively worse the longer the test was run. This problem is discussed in more detail in the problems with the automatic location detector (ALD).

### **PSD Calculations**

PSD calculations in the Post Processing Program do not use the same technique described by MITRE for the frequency domain model.(4) The PSDs are calculated by summing the squares of the Fourier coefficients (sine and cosine) at each of 200 evenly spaced frequency points from 0.1 to 20 Hz. According to the TDOP Post Processing Program Manual, all of the TDOP data were

(4) FRA Report No. FRA/ORD-78/12.III, Freight Car Truck Design Optimization, Phase I Frequency Domain Model, February 1978. filtered by 20 Hz low-pass filters, and therefore, the higher frequency points of the PSD were not calculated. The data were acquired at a rate of 200 samples per second requiring 2000 data points for a 0.1 Hz resolution. Each PSD plot uses 4000 data points in the following manner: 10 PSD calculations are made; each using 2000 points. (The first PSD calculation is made using points 1-2000, the second from 200-2200, etc.) As may be seen, 1800 points of each PSD calculated overlap those of the previous calculation. These 10 PSDs are then averaged to form one PSD for the plot. Smoothing in the frequency domain followed the summing of the PSD values. This smoothing in the frequency domain is the equivalent of applying a Hanning window in the time domain.

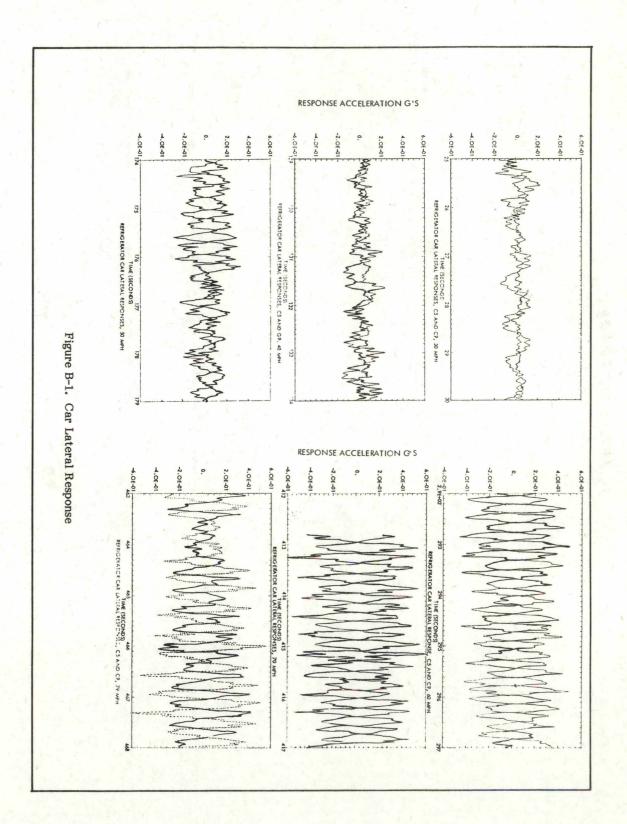
The Fourier coefficients are calculated by FORIT, an IBM scientific subroutine. This subroutine uses a variation of the direct method for calculating Fourier coefficients with the sine and cosine terms calculated recursively to reduce computational times. The subroutine allows calculation of less frequency points than what is possible, therefore, allowing only 200 frequency points to be calculated rather than the possible 1000 points. Although not nearly as efficient as an FFT routine, this method for calculating the Fourier coefficients is valid.

### Statistics

The statistical quantities calculated by the Post Processing Program are the mean value, the mean value of the rectified signal, the rms of the signal, and the standard deviation. The standard deviation calculated is not that of the rectified signal; however, there is evidence in the program listing that it was once done this way. There was controversy in the past when the SPTCo. calculated the standard deviation using the rectified signal. There was no agreement as to what the significance of the calculation was in relation to interpreting the data. This practice was apparently dropped later in Phase I and the standard deviation is now calculated using an unrectified signal. No errors were found in the calculation of these statistics. It should be noted that the rms and standard deviation of a signal with a zero mean are mathematically identical and provide no additional information. For those signals with a mean, the only significant information is derived from the standard deviation and is the value used in the analytical work.

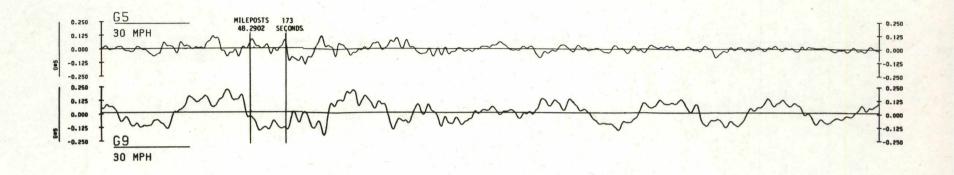
Speed	Time (Figure B-1)	Milepost (Post Processing)	Milepost (Ft)	Error	Figure
30	25	48.2902	48.2879	12'	B-2
40	129	47.2587	47.2424	861	B-3
50	174	46.7110	46.6761	184'	B-4
60	292	44.9764	44.9266	263'	B-5
79	463	41.7914	41.7006	479'	B-6

Table B-3. Time-History Comparison

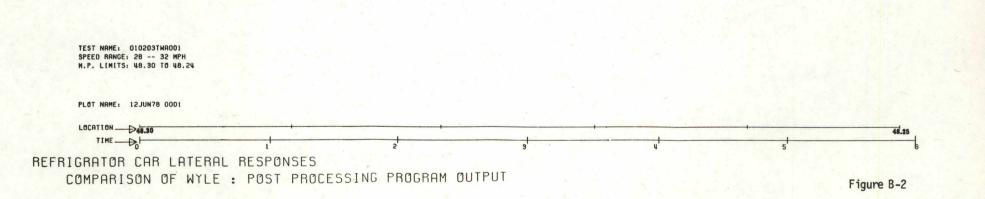


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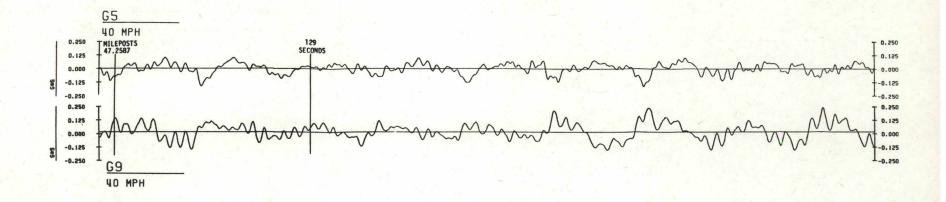
B-2







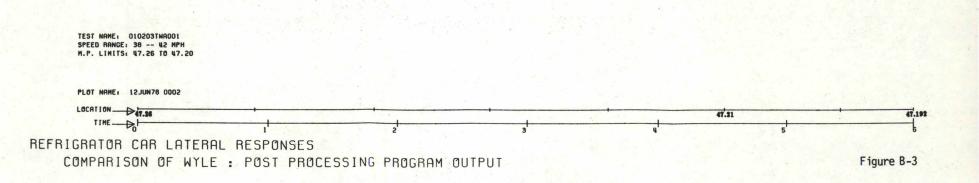
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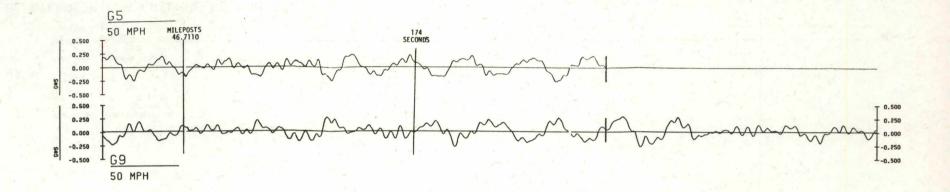


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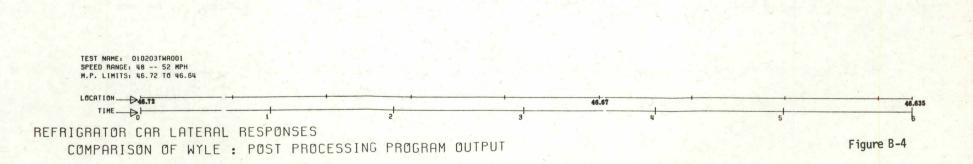
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B-8



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